BACKSCATTER FROM A SCALE-SIMILARITY MODEL: EMBEDDED LES OF CHANNEL FLOW, DEVELOPING BOUNDARY LAYER FLOW AND BACKSTEP FLOW [2] LARS DAVIDSON

Lars Davidson, www.tfd.chalmers.se/~lada

◆□▶ ◆圖▶ ◆臣▶ ◆臣▶ 臣 のへで

EMBEDDED LES: PROBLEM FORMULATION



• At the interface between RANS and LES, turbulent fluctuations, u', v', w', are imposed to stimulate growth of resolved fluctuations

EMBEDDED LES: PROBLEM FORMULATION



- At the interface between RANS and LES, turbulent fluctuations, u', v', w', are imposed to stimulate growth of resolved fluctuations
- To promote transition from RANS to LES (reducing the gray area), additional forcing may be used in the LES region

EMBEDDED LES: PROBLEM FORMULATION



- At the interface between RANS and LES, turbulent fluctuations, u', v', w', are imposed to stimulate growth of resolved fluctuations
- To promote transition from RANS to LES (reducing the gray area), additional forcing may be used in the LES region
- In the present work, forcing is added using a scale-similarity model

CHALMERS

MOMENTUM EQUATION

The momentum equations for LES read

$$\frac{D\bar{u}_i}{Dt} + \frac{1}{\rho}\frac{\partial\bar{p}}{\partial x_i} = \frac{\partial}{\partial x_k}\left((\nu + \nu_{SGS})\frac{\partial\bar{u}_i}{\partial x_k}\right) - \frac{\partial\tau_{ik}}{\partial x_k}$$

where D/Dt denotes material derivative. The stress tensor, τ_{ik} , is obtained from the scale-similarity model

$$\tau_{ik} = \overline{\bar{u}_i \bar{u}_k} - \overline{\bar{u}}_i \,\overline{\bar{u}}_k$$

CHALMERS

TSFP8, Poitiers, 2013

3 / 22

TURBULENT KINETIC ENERGY EQ

• Let us take a closer look at the equation for the resolved, turbulent kinetic energy, $\mathcal{K} = \langle \bar{u}'_i \bar{u}'_i \rangle / 2$, which reads ($\langle . \rangle$ denotes averaging in time)

$$\frac{DK}{Dt} + \langle \bar{u}'_k \bar{u}'_i \rangle \frac{\partial \langle \bar{u}_i \rangle}{\partial x_k} + \frac{1}{\rho} \frac{\partial \langle \bar{p}' \bar{u}'_i \rangle}{\partial x_i} + \frac{1}{2} \frac{\partial \langle \bar{u}'_k \bar{u}'_i \bar{u}'_i \rangle}{\partial x_k} = \nu \left\langle \frac{\partial^2 \bar{u}'_i}{\partial x_k \partial x_k} \bar{u}'_i \right\rangle - \left\langle \left(\frac{\partial \tau_{ik}}{\partial x_k} - \left\langle \frac{\partial \tau_{ik}}{\partial x_k} \right\rangle \right) \bar{u}'_i \right\rangle$$

• The second line is simply the \bar{u}'_i eq. multiplied by \bar{u}'_i

• The right side can be re-written as



The first term on the left side is the non-isotropic (i.e. the true) viscous dissipation, ε^{non}; this is predominately negative.

• The right side can be re-written as



- The first term on the left side is the non-isotropic (i.e. the true) viscous dissipation, ε^{non}; this is predominately negative.
- The first term on the right side is the viscous diffusion

• The right side can be re-written as



- The first term on the left side is the non-isotropic (i.e. the true) viscous dissipation, ε^{non}; this is predominately negative.
- The first term on the right side is the viscous diffusion
- the second term, ε , is the (isotropic) dissipation which is positive

• The right side can be re-written as



- The first term on the left side is the non-isotropic (i.e. the true) viscous dissipation, ε^{non}; this is predominately negative.
- The first term on the right side is the viscous diffusion
- the second term, ε , is the (isotropic) dissipation which is positive
- The last term, ε_{SGS}, can be positive (forward scattering=dissipation) or negative (backward scattering=forcing).

CHALMERS

PHYSICAL INTERPRETATION

The SGS term

$$\varepsilon_{SGS} = \left\langle \frac{\partial \tau_{ik}}{\partial x_k} \bar{u}'_i \right\rangle$$

consists of a net SGS force vector, T_i^{SGS} , (per unit mass), multiplied by a velocity fluctuation vector, \bar{u}'_i i.e.

$$\varepsilon_{SGS} = \left\langle T_i^{SGS} \bar{u}_i' \right\rangle$$

• When the SGS vector, T_i^{SGS} , opposes the fluctuation, \bar{u}'_i , it is damping the fluctuation, i.e. it is dissipative

Select Forward or Backscatter

We want to be able to make the term ε_{SGS} dissipative or forcing

$$\underbrace{\nu\left\langle \frac{\partial^{2}\bar{u}_{i}'}{\partial x_{k}\partial x_{k}}\bar{u}_{i}'\right\rangle}_{\varepsilon^{non}} - \left\langle \frac{\partial \tau_{ik}}{\partial x_{k}}\bar{u}_{i}'\right\rangle = \\\nu\frac{\partial^{2}K}{\partial x_{k}\partial x_{k}} - \underbrace{\nu\left\langle \frac{\partial\bar{u}_{i}'}{\partial x_{k}}\frac{\partial\bar{u}_{i}'}{\partial x_{k}}\right\rangle}_{\varepsilon} - \underbrace{\left\langle \frac{\partial \tau_{ik}}{\partial x_{k}}\bar{u}_{i}'\right\rangle}_{\varepsilon_{SGS}}$$

- The viscous term in the mom. eq., $\nu \left\langle \frac{\partial^2 \bar{u}'_i}{\partial x_k \partial x_k} \right\rangle$, is dissipative
- If $-\frac{\partial \tau_{ik}}{\partial x_k}$ has the same sign as $\frac{\partial^2 \bar{u}'_i}{\partial x_k \partial x_k}$, then ε_{SGS} is dissipative • Otherwise, it is a forcing term (backscatter)

Select Backscatter Events

- We want the SGS stress tensor to act as backscatter in the *K* equation.
- Hence we add $-\partial \tau_{ik}/\partial x_k$ to the momentum equation only when its sign is opposite to that of the viscous diffusion term. i.e. [1]

$$M_{ik} = \operatorname{sign}\left(\frac{\partial \tau_{ik}}{\partial x_k} \frac{\partial^2 \bar{u}'_i}{\partial x_k \partial x_k}\right), \quad \widetilde{M}_{ik} = \max(M_{ik}, 0), \quad \left(\frac{\partial \tau_{ik}}{\partial x_k}\right)^- = -\widetilde{M}_{ik} \frac{\partial \tau_{ik}}{\partial x_k}$$

CHALMERS

$$\frac{\partial^2 \bar{u}'_i}{\partial x_k \partial x_k} \text{ vs. } \frac{\partial^2 \bar{u}_i}{\partial x_k \partial x_k}$$

$$M_{ik} = \operatorname{sign}\left(\frac{\partial \tau_{ik}}{\partial x_k} \frac{\partial^2 \bar{u}'_i}{\partial x_k \partial x_k}\right), \quad \widetilde{M}_{ik} = \max(M_{ik}, 0), \quad \left(\frac{\partial \tau_{ik}}{\partial x_k}\right)^- = -\widetilde{M}_{ik} \frac{\partial \tau_{ik}}{\partial x_k}$$

- \bar{u}'_i , is not known at run-time. It could be computed as $\bar{u}'_i = \bar{u}_i \langle \bar{u}_i \rangle_{ra}$, where $\langle \bar{u}_i \rangle_{ra}$ denotes the running-time average of \bar{u}_i .
- It was shown in [1] that, for $y^+ \gtrsim 20$ in channel flow, the second derivative of \bar{u}'_i is almost 100% correlated with that of \bar{u}_i
- Hence, in the present work, the relation at the top-left is replaced by

$$M_{ik} = \operatorname{sign}\left(\frac{\partial \tau_{ik}}{\partial x_k} \frac{\partial^2 \bar{u}_i}{\partial x_k \partial x_k}\right)$$

STABILITY

- The forcing has a positive feedback, i.e. the more the momentum eq is destabilized, the larger the velocity gradients, the larger the forcing
- Hence, the forcing term has to be limited

$$\left|-\frac{\partial \tau_{ik}}{\partial x_k}\right| \leq \beta(\nu + \nu_{SGS}) \left|\frac{\partial^2 \bar{u}_i}{\partial x_k \partial x_k}\right|$$

The baseline value is $\beta = 2$.

PANS LOW REYNOLDS NUMBER MODEL [3]

$$\begin{split} \frac{\partial k}{\partial t} &+ \frac{\partial (kU_j)}{\partial x_j} = \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_t}{\sigma_{ku}} \right) \frac{\partial k}{\partial x_j} \right] + (P - \varepsilon) \\ \frac{\partial \varepsilon}{\partial t} &+ \frac{\partial (\varepsilon U_j)}{\partial x_j} = \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_t}{\sigma_{\varepsilon u}} \right) \frac{\partial \varepsilon}{\partial x_j} \right] + C_{\varepsilon 1} P \frac{\varepsilon}{k} - C_{\varepsilon 2}^* \frac{\varepsilon^2}{k} \\ \nu_t &= C_\mu f_\mu \frac{k^2}{\varepsilon}, C_{\varepsilon 2}^* = C_{\varepsilon 1} + \frac{f_k}{f_\varepsilon} (C_{\varepsilon 2} f_2 - C_{\varepsilon 1}), \sigma_{ku} \equiv \sigma_k \frac{f_k^2}{f_\varepsilon}, \sigma_{\varepsilon u} \equiv \sigma_\varepsilon \frac{f_k^2}{f_\varepsilon} \end{split}$$

- LRN Damping functions, f_2 , f_μ as in [3]
- RANS region: $f_k = 1.0$
- LES region: i) $f_k = 0.4 \text{ or}$ ii) $f_k = \frac{1}{c_{t_k}^{1/2}} (\Delta/L_t)^{2/3}, L_t = (k_{res} + k)^{3/2}/\varepsilon$
- Option i and ii give same results. but Option ii unstable in backstep flow with forcing

TEST CASE I: CHANNEL FLOW





3

TEST CASE I: CHANNEL FLOW





3

TEST CASE I: CHANNEL FLOW



TEST CASE II: BOUNDARY LAYER FLOW



CHALMERS

TEST CASE II: BOUNDARY LAYER FLOW







3

4 AP

TEST CASE II: BOUNDARY LAYER FLOW



INLET TURB. FLUCTUATION, 2-POINT CORRELATIONS



CHALMERS

TSFP8, Poitiers, 2013 14 / 22

RESULTS: SKIN FRICTION

RANS-LES interface



CHALMERS

Results: Resolved Shear Stresses



CHALMERS

- x = 1.25: with markers
- x = 3: without markers
- backscatter
- --- no backscatter.

BACKSTEP FLOW, COMPUTATIONAL DOMAIN

• $Re_H = 28\,000$, $336 \times 152 \times 64$ cells (x, y, z), $z_{max} = 1.6$





3

BACKSTEP FLOW, COMPUTATIONAL DOMAIN

• $Re_H = 28\,000$, $336 \times 152 \times 64$ cells (x, y, z), $z_{max} = 1.6$



3

17 / 22

Skin friction and St number



CHALMERS

TSFP8, Poitiers, 2013

18 / 22

backscatter

- - no backscatter
- o: Experiments by Vogel & Eaton [4]

CONCLUSIONS

- The gray area issue at RANS-LES interface has been addressed
- The stresses, τ_{ik} , from a scale-similarity model was used for forcing
- The forcing was achieved be selecting the instants when $-\frac{\partial T_{ik}}{\partial x_k}$ corresponds to backscatter
- It is found that the forcing indeed quickens the transition from RANS mode to LES mode
- The present approach can also be used for laminar-turbulent transition

THREE-DAY CFD COURSE AT CHALMERS

- Unsteady Simulations for Industrial Flows: LES, DES, hybrid LES-RANS and URANS
- 6-8 November 2013 at Chalmers, Gothenburg, Sweden
- Max 16 participants
- 50% lectures and 50% workshops in front of a PC
- Registration deadline: 18 October 2013
- For info, see http://www.tfd.chalmers.se/~lada/cfdkurs/cfdkurs.html

References I

[1] DAVIDSON, L.

Hybrid LES-RANS: back scatter from a scale-similarity model used as forcing.

Phil. Trans. of the Royal Society A 367, 1899 (2009), 2905–2915.

[2] DAVIDSON, L.

Backscatter from a scale-similarity model: embedded les of channel flow and developing boundary layer flow.

In 8th International Symposium on Turbulence and Shear Flow Phenomena (TSFP-8) http://www.tfd.chalmers.se/~lada/allpaper.html (Poitiers, France, 2013).

[3] MA, J., PENG, S.-H., DAVIDSON, L., AND WANG, F.

A low Reynolds number variant of Partially-Averaged Navier-Stokes model for turbulence.

International Journal of Heat and Fluid Flow 32, 3 (2011), 652-669.

A B F A B F

References II

[4] VOGEL, J., AND EATON, J.

Combined heat transfer and fluid dynamic measurements downstream a backward-facing step.

Journal of Heat Transfer 107 (1985), 922–929.