Two-equation hybrid RANS-LES Models: A novel way to treat k and ω at the inlet [2] Lars Davidson

Lars Davidson, www.tfd.chalmers.se/~lada

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 - 1.1 How do I prescribe inlet values on k and ω ?
 - 1.2 What about the URANS region? Should I prescribe k and ω from a steady RANS solution?
- 2. The proposed method is to add commutation terms in the k and ω equations.
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 equation: $-\frac{\partial \Delta}{\partial x_1} \frac{\partial \bar{u}_1 k}{\partial \Delta}$ (sink term)
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4. The method can also be used in embedded LES (i.e. at the RANS-LES interface)

▶ In the LES region, the model reads

$$\begin{aligned} \frac{\partial k}{\partial t} &+ \frac{\partial \bar{v}_i k}{\partial x_i} = P^k - f_k \frac{k^{3/2}}{\ell_t} + \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_t}{\sigma_k} \right) \frac{\partial k}{\partial x_j} \right] \\ \frac{\partial \omega}{\partial t} &+ \frac{\partial \bar{v}_i \omega}{\partial x_i} = C_{\omega_1} f_\omega \frac{\omega}{k} P^k - C_{\omega_2} \omega^2 + \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_t}{\sigma_\omega} \right) \frac{\partial \omega}{\partial x_j} \right] + C_\omega \frac{\nu_t}{k} \frac{\partial k}{\partial x_j} \frac{\partial k}{\partial$$

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- The length scale, Δ_{dw}, is taken from the IDDES model [9].
- In the RANS regions, $\ell_t = k^{1/2}/(C_k \omega)$.
- ► The interface between LES and RANS regions is chosen at a fixed grid line (y⁺ ≃ 500)

VARYING FILTER SIZE

- When filter size in LES varies in space, an additional term appears in the momentum equation.
- The reason? the spatial derivatives and the filtering do not commute.

For the convective term in Navier-Stokes, for example, we get $\overline{\frac{\partial v_i v_j}{\partial x_j}} = \frac{\partial}{\partial x_j} (\overline{v_i v_j}) + \mathcal{O}((\Delta x)^2)$

Ghosal & Moin [4] showed that the error is proportional to (Δx)²; hence it is usually neglected.

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Commutation error in k equation

- In zonal¹ hybrid RANS-LES, the length scale at the RANS-LES interface changes abruptly from a RANS length scale to a LES length scale.
- Hamda [5] found that the commutation error at RANS-LES interfaces is large.
- For the k equation the commutation term reads

$$\overline{\frac{\partial u_i k}{\partial x_i}} = \frac{\partial \bar{u}_i k}{\partial x_i} - \frac{\partial \Delta}{\partial x_i} \frac{\partial \bar{u}_i k}{\partial \Delta}$$

COMMUTATION TERM: PHYSICAL MEANING

$$\overline{\frac{\partial u_i k}{\partial x_i}} = \frac{\partial \bar{u}_i k}{\partial x_i} - \frac{\partial \Delta}{\partial x_i} \frac{\partial \bar{u}_i k}{\partial \Delta}$$

Consider a fluid particle in a RANS region moving in the x₁ direction and passing across a RANS-LES interface.

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► *k* decreases when going from RANS to LES ⇒ $\partial \bar{u}_1 k / \partial \Delta = \underbrace{(k_{LES} - k_{RANS})}_{<0} / \underbrace{(\Delta_{LES} - \Delta_{RANS})}_{<0} > 0$

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 $\blacktriangleright \Rightarrow The commutation term > 0$

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- Consider a fluid particle in a RANS region moving in the x₁ direction and passing across a RANS-LES interface.
- The filterwidth decreases across the interface, i.e. ∂∆/∂x₁ < 0</p>
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- $\blacktriangleright \Rightarrow The commutation term > 0$
- ► ⇒ The commutation term < 0 on the right-side of the k equation.</p>

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- Consider a fluid particle in a RANS region moving in the x₁ direction and passing across a RANS-LES interface.
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 \blacktriangleright \Rightarrow The commutation term > 0

- ► ⇒ The commutation term < 0 on the right-side of the k equation.</p>
- Hence, the commutation term at the RANS-LES interface reduces k.

Commutation term at the RANS-LES INTERFACE



Commutation term at the RANS-LES INTERFACE



COMMUTATION TERM AT THE LES INLET



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COMMUTATION TERM AT THE LES INLET



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Commutation term in the ω equation

• Let us start by looking at the ε equation.

- What happens with ε when a fluid particle moves from a RANS region into an LES region?
- The answer is, nothing. The dissipation is the same in a RANS region as in an LES region.
- \blacktriangleright Transformation of the k and ε equations to an ω equation gives

$$\frac{d\omega}{dt} = \frac{d}{dt} \left(\frac{\varepsilon}{C_k k}\right) = \frac{1}{C_k k} \frac{d\varepsilon}{dt} + \frac{\varepsilon}{C_k} \frac{d(1/k)}{dt} = \frac{1}{C_k k} \frac{d\varepsilon}{dt} - \frac{\omega}{k} \frac{dk}{dt}$$

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Hence, the commutation error in the ω equation is the commutation term in the k equation multiplied by -ω/k so that

$$\overline{\frac{\partial u_i \omega}{\partial x_i}} = \frac{\partial \overline{u}_i \omega}{\partial x_i} - \frac{\partial \Delta}{\partial x_i} \frac{\partial \overline{u}_i \omega}{\partial \Delta} = \frac{\partial \overline{u}_i \omega}{\partial x_i} + \frac{\omega}{k} \frac{\partial \Delta}{\partial x_i} \frac{\partial \overline{u}_i k}{\partial \Delta}$$

Prescribe RANS values of k and ω at the inlet obtained from RANS simulations

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 equation: $\frac{\omega}{k} \frac{\partial \Delta}{\partial x_1} \frac{\partial \overline{u}_1 k}{\partial \Delta}$ (source term)

 The present approach is similar to adding the commutation term in PANS [3]

$$f_{k}\frac{Dk_{tot}}{Dt} = \frac{D(f_{k}k_{tot})}{Dt} - k_{tot}\frac{Df_{k}}{Dt} = \frac{Dk}{Dt} - \frac{k_{tot}}{Dt}\frac{Df_{k}}{Dt}$$
$$\frac{D}{Dt} = \frac{\partial}{\partial t} + \bar{u}_{i}\frac{\partial}{\partial x_{i}}, \quad k_{tot} = k + \frac{1}{2}\langle u_{i}'u_{i}'\rangle$$

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we can only use the Reynolds stress tensor in one point

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 - we can only use the Reynolds stress tensor in one point
 - We need to chose a relevant location for the Reynolds stress tensor
 - In boundary layer flow, the turbulent shear stress is the single most important stress component

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- 2. The Reynolds stress tensor is computed using the EARSM model [10].
- 3. Synthetic turbulence fluctuations based on homogeneous turbulence
 - we can only use the Reynolds stress tensor in one point
 - We need to chose a relevant location for the Reynolds stress tensor
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- 4. Finally, the synthetic fluctuations are scaled with $(|\overline{u'v'}|/|\overline{u'v'}|_{max})_{RANS}^{1/2}$ which is taken from the RANS simulation.
Synthetic inlet fluctuations

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- 2. The Reynolds stress tensor is computed using the EARSM model [10].
- 3. Synthetic turbulence fluctuations based on homogeneous turbulence
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 - In boundary layer flow, the turbulent shear stress is the single most important stress component
 - Hence, the Reynolds stress tensor is taken at the location where the magnitude of the turbulent shear stress is largest.
- 4. Finally, the synthetic fluctuations are scaled with $(|\overline{u'v'}|/|\overline{u'v'}|_{max})_{RANS}^{1/2}$ which is taken from the RANS simulation.
- 5. Matlab codes can be downloaded [1] (Google "synthetic inlet fluctuations")

CHANNEL FLOW

- Reynolds number is $Re_{\tau} = 8000$.
- A 256 \times 96 \times 32 mesh is used
- $\Delta x = 0.1, \ \Delta z = 0.05$
- ▶ The mean *U*, *k* and ω taken from 1D RANS simulation using the PDH *k* − ω model
- ► The wall-parallel RANS-LES interface is prescribed at a fixed gridline at y⁺ ≃ 500.



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INLET FLUCTUATIONS



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RESULTS



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RESULTS



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LENGTH OF SOURCE REGION

In how large a region, x_{tr}, should the commutation terms be added?



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RESULTS



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Source terms in k equation



 $x/\delta = 0.05.$ o: Production term, P^k , $x_{tr} = 0.$ —: commutation term. $(x_{tr}/\delta = 0.1, 0.5, 1)$ Arrow shows increasing x_{tr}

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Commutation terms in the (U)RANS region?



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Commutation terms in the (U)RANS region?



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Commutation terms in the (U)RANS region?



Argument for using commutation terms in the (U)RANS region: ν_{t,URANS} ≪ ν_{t,RANS}

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Commutation term or not in (U)RANS region?

BLUE LINES: commutation terms in the (U)RANS region RED LINES: no commutation terms in the (U)RANS region



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solid lines: $x/\delta_{in} = 0.05$ dashed lines: $x/\delta_{in} = 2.5$

CONCLUSIONS

- A novel method for prescribing inlet modelled turbulent quantities (k, ε, ω) has been presented
- It is based on the non-commutation between the divergence and the filter operators
- No tuning constants
- It is best to impose the commutation terms in one grid plane adjacent to the inlet

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THREE-DAY CFD COURSE AT CHALMERS

- Unsteady Simulations for Industrial Flows: LES, DES, hybrid LES-RANS and URANS
- ▶ 9-11 November 2015 at Chalmers, Gothenburg, Sweden
- Max 16 participants
- ▶ 50% lectures and 50% workshops in front of a PC
- Registration deadline: 10 October 2014
- For info, see

http://www.tfd.chalmers.se/~lada/cfdkurs/cfdkurs.html

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