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# Inlet boundary conditions for Hybrid LES-RANS (Davidson, 2007a, 2007b, 2007c)

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#### **HYBRID LES-RANS**

**Near walls:** a RANS one-eq. k model. In core region: a LES one-eq.  $k_{SGS}$  model.



# **MOMENTUM EQUATIONS**

• The Navier-Stokes, time-averaged in the near-wall regions and filtered in the core region, reads

$$\frac{\partial \bar{U}_i}{\partial t} + \frac{\partial}{\partial x_j} \left( \bar{U}_i \bar{U}_j \right) = \beta \delta_{1i} - \frac{1}{\rho} \frac{\partial \bar{P}}{\partial x_i} + \frac{\partial}{\partial x_j} \left[ (\nu + \nu_T) \frac{\partial \bar{U}_i}{\partial x_j} \right]$$
$$\nu_T = \nu_t, y \le y_{ml}; \nu_T = \nu_{sgs}, y \ge y_{ml}$$

0 10

#### **TURBULENCE MODEL**

• Use one-equation model in both URANS region and LES region.

$$\begin{aligned} \frac{\partial k_T}{\partial t} + \frac{\partial}{\partial x_j} (\bar{u}_j k_T) &= \frac{\partial}{\partial x_j} \left[ (\nu + \nu_T) \frac{\partial k_T}{\partial x_j} \right] + P_{k_T} - C_{\varepsilon} \frac{k_T^{3/2}}{\ell} \\ P_{k_T} &= 2\nu_T \bar{S}_{ij} \bar{S}_{ij}, \ \nu_T = C_k \ell k_T^{1/2} \end{aligned}$$

**LES-region:**  $k_T = k_{sgs}$ ,  $\nu_T = \nu_{sgs}$ ,  $\ell = \Delta = (\delta V)^{1/3}$ **URANS-region:**  $k_T = k$ ,  $\nu_T = \nu_t$ ,  $\ell \propto y$ , Chen-Patel model (AIAA J. 1988)



•  $[\kappa_1, \kappa_{max}]$  divided into N modes,  $\kappa_{max} = \frac{\pi}{\min\{\Delta x, \Delta y, \Delta z\}}, \kappa_1 = \kappa_e/2$ 

#### TIME SCALES

• *M* independent realizations  $u_t(x)$  are created. Thus no time correlation. A time correlation is introduced by

 $(\mathcal{U}')^m = a(\mathcal{U}')^{m-1} + b(\mathbf{u}_t(\mathbf{x}))^m, m = \text{time step}$ 

autocorrelation  $B(\tau)$ 0.8 The autocorrelation 0.6  $B(\tau)$ is prescribed by setting 04  $a = \exp(-\Delta t/T), b = (1 - a^2)^{1/2}$ 0.2  $---- \exp(-\tau/\mathcal{T})$ 0 -----  $B(\tau)$  from  $(\mathcal{U}')^m$ 0.2 0.4 0.8 0.6 0  $\mathcal{T}$ 

#### **CHANNEL WITH INLET-OUTLET**

• Inlet fluctuations are set as  $(\mathcal{U}')^m$ ,  $(\mathcal{V}')^m$ ,  $(\mathcal{W}')^m$ .

• The streamwise fluctuations are superimposed to the mean profile

$$U_{in}^{+} = \begin{cases} y^{+} & y^{+} \leq 5\\ -3.05 + 5\ln(y^{+}) & 5 < y^{+} < 30\\ \frac{1}{\kappa}\ln(y^{+}) + B & y^{+} \geq 30 \end{cases}$$

where  $\kappa = 0.4$  and B = 5.2



•  $64 \times 64 \times 32$  (*x*, *y*, *z*) cells.  $z_{max} = 6.3\delta$ ,  $\Delta x^+ \simeq 785$ ,  $\Delta z^+ \simeq 393$ .

- $\delta/\Delta z \simeq 5$ ,  $\delta/\Delta x \simeq 2.5$
- The location of the matching plane at y = 0.075 ( $y^+ = 150$ )

#### TEST CASES

- Different inlet turbulent length and time scales have been used
- Using k = 4,  $\varepsilon = 70$  (scaled with  $u_{\tau}$  and  $\delta$ ), the baseline scales were set to:

Length scale:  $L_t = k^{3/2}/\varepsilon = \mathcal{L}_1 = 0.11\delta$ 

Time scale:  $\tau_t = 4k/\varepsilon = \mathcal{T}_1 = 0.22\delta/u_{\tau}$ 

**Time scale**  $\mathcal{T}_1$  **and length scale**  $\mathcal{L}_1$ 



**Time scale**  $T_1$  **and length scale**  $2L_1$ 



**Time scale**  $T_1$  **and length scale**  $L_t = 0$ 



**Time scale**  $\tau = 0$  **and length scale**  $\mathcal{L}_1$ 



**Time scale**  $0.25\mathcal{T}_1$  and length scale  $\mathcal{L}_1$ 



**FRICTION VELOCITIES** 



# FORCING FLUCTUATIONS ADDED AT THE INTERFACE

• Object: to trig the momentum equations into resolving largescale turbulence

interface



# **INLET BOUNDARY CONDITIONS vs. FORCING**

Inlet



#### **FULLY DEVELOPED CHANNEL FLOW (periodic in** *x***)**



+: 
$$U^+ = \frac{1}{0.4} \ln y^+ + 5.2$$

— no forcing

---- forcing (isotropic fluctuations)

# • $\mathcal{T}_1, \mathcal{L}_1$



# **FRICTION VELOCITIES**, X-LONG CHANNEL • $T_1$ , $L_1$







#### • Velocities

#### Lars Davidson

#### **CHALMERS**



**CHALMERS** 



**3D HILL** 



—— fluctating inlet b.c. ------ steady inlet b.c.



- 1. Do a RANS simulation of the entire domain
- 2. Do an unsteady LES/RANS simulation of the right part of the domain
- At stage 2, fluctuating inlet boundary conditions needed

# **FLOW OVER a BUMP: DESider CASE**

• Measurements have been carried out by ONERA.  $Re_h = 2.1 \cdot 10^6$  based on the bump height. Rectangular duct.

• W/H = 1.67, h/H = 0.46,  $L_1/H = 0.34$ ,  $L_2/H = 0.88$ , L/H = 7.6,  $\delta_{in}/H = 0.03$ .

• Mesh:  $221 \times 122 \times 32$ . Note that  $\delta_{in}/\Delta z \simeq 2$  and  $\delta_{in}/\Delta x_{in} \simeq 3$ 



#### **FLOW OVER a BUMP: shear stresses**



- —— resolved shear stresses; fluctating inlet b.c.
- ----- resolved shear stresses; steady inlet b.c.
- ----- modelled shear stresses; fluctating inlet b.c.

## FLOW OVER a BUMP: pressure coefficient



# CONCLUSIONS

• Synthesized isotropic turbulence fluctuations have been used for prescribing fluctuating inlet velocities.

- Length scale  $\mathcal{L}_1$  and time scale  $\mathcal{T}$  prescribed independently
- It has been shown that both  $\mathcal{L}_1$  and  $\mathcal{T}$  are essential

• Far downstream the inlet, the standard hybrid LES-RANS forgets the inlet turbulence

• Hybrid LES-RANS with forcing conditions are needed for long channels

• Forcing conditions similar to inlet boundary conditions of fluctuating velocities. The object is to trig the equations into resolving turbulence.

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