# A New Approach to Treat the RANS-LES INTERFACE IN PANS [1] LARS DAVIDSON

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**PANS** Low Reynolds Number Model [4]

$$\begin{split} \frac{\partial k}{\partial t} &+ \frac{\partial (kU_j)}{\partial x_j} = \frac{\partial}{\partial x_j} \left[ \left( \nu + \frac{\nu_t}{\sigma_{ku}} \right) \frac{\partial k}{\partial x_j} \right] + (P - \varepsilon) \\ \frac{\partial \varepsilon}{\partial t} &+ \frac{\partial (\varepsilon U_j)}{\partial x_j} = \frac{\partial}{\partial x_j} \left[ \left( \nu + \frac{\nu_t}{\sigma_{\varepsilon u}} \right) \frac{\partial \varepsilon}{\partial x_j} \right] + C_{\varepsilon 1} P \frac{\varepsilon}{k} - C_{\varepsilon 2}^* \frac{\varepsilon^2}{k} \\ \nu_t &= C_\mu f_\mu \frac{k^2}{\varepsilon}, C_{\varepsilon 2}^* = C_{\varepsilon 1} + \frac{f_k}{f_\varepsilon} (C_{\varepsilon 2} f_2 - C_{\varepsilon 1}), \sigma_{ku} \equiv \sigma_k \frac{f_k^2}{f_\varepsilon}, \sigma_{\varepsilon u} \equiv \sigma_\varepsilon \frac{f_k^2}{f_\varepsilon} \end{split}$$

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- LRN Damping functions,  $f_2$ ,  $f_\mu$  as in [4]
- $f_{\varepsilon} = 1.0$
- LES region:  $f_k = 0.4$
- RANS region:  $f_k = 1.0$

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>Zonal RANS-LES:  $f_k$  has a large gradient at the RANS-LES interface

## PANS: DERIVATION

• The PANS *k* equation is derived by multiplying the RANS *k* equation by *f<sub>k</sub>*. The left hand reads

$$f_k \frac{Dk_{tot}}{Dt} \tag{1}$$

where  $D/Dt = \partial/\partial t + \bar{u}_i \partial/\partial x_i$ ,  $k_{tot} = k + k_{res}$  (modeled plus resolved).

• If it is assumed that  $f_k$  constant, Eq. 1 can be re-written as

$$f_k \frac{Dk_{tot}}{Dt} = \frac{Df_k k_{tot}}{Dt} = \frac{Dk}{Dt}, \quad f_k = \frac{k}{k_{tot}}$$
(2)

If f<sub>k</sub> is not constant, Eq. 2 must be written as (Girimaji & Wallin [2])

$$f_k \frac{Dk_{tot}}{Dt} = \frac{Df_k k_{tot}}{Dt} - k_{tot} \frac{Df_k}{Dt} = \frac{Dk}{Dt} - \frac{k_{tot} \frac{Df_k}{Dt}}{Dt}$$

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- This work presents models for the boxed term at RANS-LES interfaces, i.e.
  - horizontal RANS-LES interface in boundary layer (channel flow)
  - vertical RANS-LES interface in embedded LES (channel flow)

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## INTERFACE MODEL 1

• In [2],  $k_{tot}Df_k/Dt$  is represented by introducing an additional turbulent viscosity,  $\nu_{tr}$ , in the momentum equation

$$\frac{\partial}{\partial x_j} \left( \nu_{tr} \bar{s}_{ij} \right), \quad \bar{s}_{ij} = \frac{1}{2} \left( \frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right)$$

where

$$\nu_{tr} = \frac{P_{k_{tr}}}{|\bar{s}|^2}, \quad P_{k_{tr}} = \nu_{tr}|\bar{s}|^2 = k_{tot}\frac{Df_k}{Dt} = \frac{k}{f_k}\frac{Df_k}{Dt}$$

- The object of  $P_{k_{tr}}$  is to decrease  $\nu_t$  and facilitate growth of resolved turbulence on the LES side of an interface
- Hence, only  $\nu_{tr} < 0$  is used which corresponds to  $Df_k/Dt < 0$  (from RANS to LES).
- $\nu_t + \nu_{tr} > 0$  in the momentum equation (but not in the k equation)

## INTERFACE MODEL 2

• This model is identical to Model 1 except that  $k/f_k$  is replaced by  $k_{tot}$  i.e.

$$\frac{P_{k_{tr}} = \frac{k}{f_k} \frac{Df_k}{Dt}}{Model 1} \qquad \frac{P_{k_{tr}} = k_{tot} \frac{Df_k}{Dt}}{Model 2}}{k_{tot} = k + \frac{1}{2} \langle \bar{u}'_i \bar{u}'_i \rangle_{r.a}}$$

where subscript r.a. denotes running average.

- In PANS,  $f_k$  is defined as  $f_k = k/k_{tot}$
- In post-processing it is usually found that  $f_k > k/k_{tot}$  (approx. a factor 4 larger)  $\Rightarrow |P_{k_{tr}}|_{model 2} > |P_{k_{tr}}|_{model 1}$

## INTERFACE MODEL 3

• In Models 1 & 2,  $u_{tr} = P_{k_{tr}}/|\bar{s}|^2$  which may cause numerical problems.

- Model 3 does not involve  $\nu_{tr}$ . The original term  $k_{tot}Df_k/Dt$  is used in the k equation
- Adding the term

$$-0.5\bar{u}_{i}^{\prime}\frac{Df_{k}}{Dt}-\frac{k\bar{u}_{i}^{\prime}}{\langle\bar{u}_{m}^{\prime}\bar{u}_{m}^{\prime}\rangle}\frac{Df_{k}}{Dt}$$

in the momenum equation corresponds to the time-averaged term

$$-\left\langle \frac{\bar{u}_{i}'\bar{u}_{i}'}{2} \frac{Df_{k}}{Dt} \right\rangle - \frac{k \langle \bar{u}_{i}'\bar{u}_{i}' \rangle}{\langle \bar{u}_{m}'\bar{u}_{m} \rangle} \frac{Df_{k}}{Dt} = -\langle k_{tot} \rangle \frac{Df_{k}}{Dt}$$

in the  $k_{res}$  equation.

• However, this term causes numerical instability. Hence it is not used.

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• Only the term in the k equation,  $k_{tot}Df_k/Dt < 0$ , is used.

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## INTERFACE MODELS: SUMMARY

- Models 1 & 2: additional turbulent viscosity,  $\nu_{tr} = P_{k_{tr}}/|\bar{s}|^2 < 0$ , in  $P_k$  and momentum equations
  - Limit in momentum equations:  $v_t + v_{tr} > 0$

 Model 3: additional production term, P<sub>ktr</sub>, in k equation without use of ν<sub>tr</sub>

$$\mathsf{P}_{k_{tr}} = \langle k_{tot} \rangle_{r.a} \frac{Df_k}{Dt} < 0$$

 Models 1-3 correspond to the non-commutivity in DES beteen filtering and spatial derivative at RANS-LES interfaces (Hamba [3])

$$\frac{\overline{\partial f}}{\partial x_i} = \frac{\partial \overline{f}}{\partial x_i} - \frac{\partial \Delta}{\partial x_i} \frac{\partial \overline{f}}{\partial \Delta}$$

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FULLY DEVELOPED CHANNEL FLOW

• The URANS and the LES regions.

LES,  $f_k = 0.4$ 



•  $\textit{Re}_{ au} = \textit{u}_{ au} \delta / \nu = 2\,000$ ,  $\textit{Re} = 4\,000$  and  $\textit{Re} = 8\,000$ 

•  $x_{max} = 3.2$ ,  $y_{max} = 2$  and  $z_{max} = 1.6$ .

•  $32 \times 32$  cells in the x - z plane

•  $N_y = 80$  cells ( $Re_\tau = 2\,000$  and  $4\,000$ ) or  $N_y = 96$  ( $Re_\tau = 8\,000$ )

## Fully Developed Channel Flow: Results



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•: location of the computational cell centers.

## FULLY DEVELOPED CHANNEL FLOW: RESULTS



---: Model  $\frac{k}{f_k} \frac{Df_k}{Dt}$  --: Model  $k_{tot} \frac{Df_k}{Dt}$ . ...: no interface model. •: the location of the computational cell centers.

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## Embedded Channel Flow



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• 
$$Re_{ au} = u_{ au}\delta/
u = 950$$

- The domain size is  $6.4 \times 2 \times 1.6 (x, y, z)$
- $128 \times 80 \times 64$  cells.

## Synthetic Fluctuations at the Interface



Two-point correlation of synthetic inlet fluctuations

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## VELOCITY AND SKIN FRICTION



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## Resolved and Modeled Turbulence



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## PRODUCTION AT INTERFACE



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 $\cdots$ :  $P_k$ , no interface model;



• Three interface models for horizontal (wall-parallel) and vertical interfaces (embedded LES) have been presented:



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•  $\frac{k}{f_k} \frac{Df_k}{Dt}$  added via  $\nu_{t_{tr}}$  to the k eq and mom eq  $(\nu_{t_{tr}} + \nu_t > 0)$ : Model 1

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  - $k_{tot} \frac{Df_k}{Dt}$  added no  $\nu_{t_{tr}}$  to the k eq (Model 3)
- Model 2 works very well
- Model 3 gives identical results to Model 2 (not shown in this presentation)

## THREE-DAY CFD COURSE AT CHALMERS

- Unsteady Simulations for Industrial Flows: LES, DES, hybrid LES-RANS and URANS
- 29-31 October 2014 at Chalmers, Gothenburg, Sweden
- Max 16 participants
- 50% lectures and 50% workshops in front of a PC
- Registration deadline: 10 October 2014
- For info, see http://www.tfd.chalmers.se/~lada/cfdkurs/cfdkurs.html

# References I

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