

2017-05-30, Exam in

Turbulence modeling, MTF270

- **Time:** 08.30-12.30 **Location:** M
 - **Teacher:** Lars Davidson, phone 772 1404, 0730-791 161
 - **Aids**
 - Formula sheet, 2 pages, appended to the exam
 - **Checking the evaluation and results of your written exam:** at 12-13 on 19 and 20 June in room Navier
 - **Grading:** 20-29p: 3, 30-39: 4, 40-50: 5.
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T1. a) What is the expression for the exact total heat flux that appears in the $\bar{\theta}$ equation? (5p)

b) Consider the equation below. (5p)

$$\begin{aligned} \frac{\partial \overline{v'_i v'_j}}{\partial t} + \underbrace{\bar{v}_k \frac{\partial \overline{v'_i v'_j}}{\partial x_k}}_{C_{ij}} &= \underbrace{-\overline{v'_i v'_k} \frac{\partial \bar{v}_j}{\partial x_k} - \overline{v'_j v'_k} \frac{\partial \bar{v}_i}{\partial x_k}}_{P_{ij}} + \underbrace{\frac{p'}{\rho} \left(\frac{\partial v'_i}{\partial x_j} + \frac{\partial v'_j}{\partial x_i} \right)}_{\Pi_{ij}} \\ &\quad - \underbrace{\frac{\partial}{\partial x_k} \left[\overline{v'_i v'_j v'_k} + \frac{\overline{p' v'_j}}{\rho} \delta_{ik} + \frac{\overline{p' v'_i}}{\rho} \delta_{jk} \right]}_{D_{ij,t}} + \underbrace{\nu \frac{\partial^2 \overline{v'_i v'_j}}{\partial x_k \partial x_k}}_{D_{ij,\nu}} \\ &\quad - \underbrace{g_i \beta \overline{v'_j \theta'} - g_j \beta \overline{v'_i \theta'}}_{G_{ij}} - \underbrace{2\nu \frac{\partial v'_i}{\partial x_k} \frac{\partial v'_j}{\partial x_k}}_{\varepsilon_{ij}} \end{aligned}$$

Explain the physical meaning of the different terms. Which terms need to be modeled?

T2. a) How is the production term modeled in the $k - \varepsilon$ model? Show how it can be expressed in \bar{s}_{ij} (5p)

b) Consider the exact equation for the pressure fluctuation. (3p)

$$p'(\mathbf{x}) = \frac{\rho}{4\pi} \int_V \left[2 \frac{\partial \bar{v}_i(\mathbf{y})}{\partial y_j} \frac{\partial v'_j(\mathbf{y})}{\partial y_i} + \frac{\partial^2}{\partial y_i \partial y_j} \left(v'_i(\mathbf{y}) v'_j(\mathbf{y}) - \overline{v'_i(\mathbf{y}) v'_j(\mathbf{y})} \right) \right] \frac{d\mathbf{y}^3}{|\mathbf{y} - \mathbf{x}|}$$

Which term is the “slow” and “rapid” terms? Why are they called “slow” and “rapid”?

- T3. a) When formulating a non-linear model, the anisotropy tensor $a_{ij} = -2\nu_t \bar{s}_{ij}/k$ is often used. The first four terms read (7p)

$$a_{ij} = -2c_\mu \tau \bar{s}_{ij} + c_1 \tau^2 \left(\bar{s}_{ik} \bar{s}_{kj} - \frac{1}{3} \bar{s}_{\ell k} \bar{s}_{\ell k} \delta_{ij} \right) \\ + c_2 \tau^2 \left(\bar{\Omega}_{ik} \bar{s}_{kj} - \bar{s}_{ik} \bar{\Omega}_{kj} \right) \\ + c_3 \tau^2 \left(\bar{\Omega}_{ik} \bar{\Omega}_{jk} - \frac{1}{3} \bar{\Omega}_{\ell k} \bar{\Omega}_{\ell k} \delta_{ij} \right)$$

Show that each term has the same properties as a_{ij} , i.e. non-dimensional, traceless and symmetric.

- b) Derive a transport equation for ω from the k and ε transport equations; you only need to do the production, the destruction and the viscous diffusion terms. (5p)
- T4. a) Consider Fig. 1. A-F denotes energy transfer. Identify A-F with one of the following expression (5p)

$$-\langle \bar{v}'_i \bar{v}'_j \rangle \frac{\partial \langle \bar{v}_i \rangle}{\partial x_j}, \quad 2\langle \nu_{sgs} \rangle \langle \bar{s}_{ij} \rangle \langle \bar{s}_{ij} \rangle, \quad \nu \frac{\partial \langle \bar{v}_i \rangle}{\partial x_j} \frac{\partial \langle \bar{v}_i \rangle}{\partial x_j}, \quad \nu \left\langle \frac{\partial \bar{v}'_i}{\partial x_j} \frac{\partial \bar{v}'_i}{\partial x_j} \right\rangle, \quad \varepsilon'_{sgs}, \quad \varepsilon$$

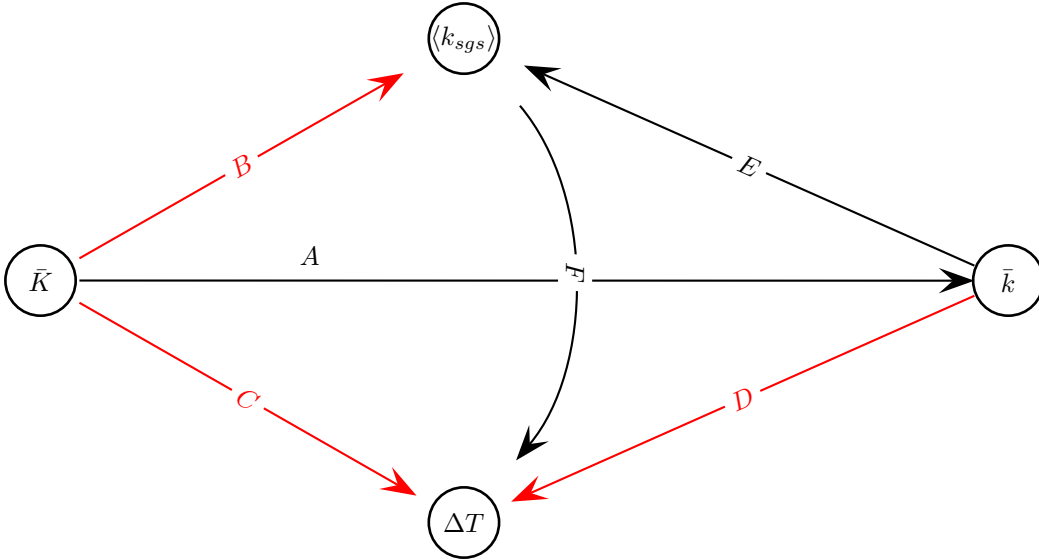


Figure 1: Transfer of kinetic turbulent energy. $\bar{K} = \frac{1}{2} \langle \bar{v}_i \rangle \langle \bar{v}_i \rangle$ and $\bar{k} = \frac{1}{2} \langle \bar{v}'_i \bar{v}'_i \rangle$ denote time-averaged kinetic and resolved turbulent kinetic energy, respectively. ΔT denotes increase in internal energy, i.e. dissipation. The cascade process assumes that the terms in red are negligible.

- b) The filtered non-linear term on the left side has the form (5p)

$$\frac{\overline{\partial v_i v_j}}{\partial x_j}$$

Show that it can be re-written as

$$\frac{\partial \bar{v}_i \bar{v}_j}{\partial x_j}$$

giving an additional term

$$-\frac{\partial \tau_{ij}}{\partial x_j}$$

on the right side.

- T5. a) What is a test filter? Grid and test filter Navier-Stokes equation and derive the relation (5p)

$$\overline{\bar{v}_i \bar{v}_j} - \widehat{\bar{v}_i \bar{v}_j} + \widehat{\tau}_{ij} = \mathcal{L}_{ij} + \widehat{\tau}_{ij} = T_{ij} \quad (1)$$

- b) The ε_u equation in the PANS model reads (5p)

$$\frac{\partial \varepsilon_u}{\partial t} + \frac{\partial (\varepsilon_u \bar{v}_j)}{\partial x_j} = \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_u}{\sigma_{\varepsilon u}} \right) \frac{\partial \varepsilon_u}{\partial x_j} \right] + C_{\varepsilon 1} P_u \frac{\varepsilon_u}{k_u} - C_{\varepsilon 2}^* \frac{\varepsilon_u^2}{k_u}$$

where

$$C_{\varepsilon 2}^* = C_{\varepsilon 1} + \frac{f_k}{f_\varepsilon} (C_{\varepsilon 2} - C_{\varepsilon 1}), \quad C_{\varepsilon 1} = 1.5, \quad C_{\varepsilon 2} = 1.9$$

Assume that $f_\varepsilon = 1$. Consider the destruction term in the ε equation and the coefficient $C_{\varepsilon 2}^*$. Explain what happens if f_k is reduced from 1 (RANS mode) to $f_k = 0.4$ (LES mode).

MTF270 Turbulence modeling: Formula sheet

March 25, 2019

The continuity, Navier-Stokes and temperature equations for incompressible flow with constant viscosity read (*conservative* form)

$$\begin{aligned}\frac{\partial v_i}{\partial x_i} &= 0 \\ \rho_0 \frac{\partial v_i}{\partial t} + \rho_0 \frac{\partial v_i v_j}{\partial x_j} &= -\frac{\partial p}{\partial x_i} + \mu \frac{\partial^2 v_i}{\partial x_j \partial x_j} - \rho_0 \beta (\theta - \theta_0) g_i \\ \frac{\partial \theta}{\partial t} + \frac{\partial v_i \theta}{\partial x_i} &= \alpha \frac{\partial^2 \theta}{\partial x_i \partial x_i}\end{aligned}$$

► The Navier-Stokes equation for incompressible flow with constant viscosity read (*non-conservative* form, p denotes the hydrostatic pressure, i.e. $p = 0$ if $v_i = 0$)

$$\rho_0 \frac{\partial v_i}{\partial t} + \rho_0 v_j \frac{\partial v_i}{\partial x_j} = -\frac{\partial p}{\partial x_i} + \mu \frac{\partial^2 v_i}{\partial x_j \partial x_j}$$

The *time averaged* continuity equation, Navier-Stokes equation temperature equations read

$$\begin{aligned}\frac{\partial \bar{v}_i}{\partial x_i} &= 0 \\ \rho_0 \frac{\partial \bar{v}_i \bar{v}_j}{\partial x_j} &= -\frac{\partial \bar{p}}{\partial x_i} + \frac{\partial}{\partial x_j} \left(\mu \frac{\partial \bar{v}_i}{\partial x_j} - \rho_0 \overline{v'_i v'_j} \right) - \rho_0 \beta (\bar{\theta} - \theta_0) g_i \\ \frac{\partial \bar{v}_i \bar{\theta}}{\partial x_i} &= \alpha \frac{\partial^2 \bar{\theta}}{\partial x_i \partial x_i} - \frac{\partial \overline{v'_i \theta'}}{\partial x_i}\end{aligned}$$

The Boussinesq assumption reads

$$\overline{v'_i v'_j} = -\nu_t \left(\frac{\partial \bar{v}_i}{\partial x_j} + \frac{\partial \bar{v}_j}{\partial x_i} \right) + \frac{2}{3} \delta_{ij} k = -2\nu_t \bar{s}_{ij} + \frac{2}{3} \delta_{ij} k$$

The modeled $\overline{v'_i v'_j}$ equation with IP model reads

$$\begin{aligned}\bar{v}_k \frac{\partial \overline{v'_i v'_j}}{\partial x_k} &= \text{(convection)} \\ -\overline{v'_i v'_k} \frac{\partial \bar{v}_j}{\partial x_k} - \overline{v'_j v'_k} \frac{\partial \bar{v}_i}{\partial x_k} &= \text{(production)} \\ -c_1 \frac{\varepsilon}{k} \left(\overline{v'_i v'_j} - \frac{2}{3} \delta_{ij} k \right) &= \text{(slow part)} \\ -c_2 \left(P_{ij} - \frac{2}{3} \delta_{ij} P^k \right) &= \text{(rapid part)} \\ +c_{1w} \rho_0 \frac{\varepsilon}{k} \left[\overline{v'_k v'_m} n_k n_m \delta_{ij} - \frac{3}{2} \overline{v'_i v'_k} n_k n_j \right. \\ &\quad \left. - \frac{3}{2} \overline{v'_j v'_k} n_k n_i \right] f \left[\frac{\ell_t}{x_n} \right] &= \text{(wall, slow part)} \\ +c_{2w} \left[\Phi_{km,2} n_k n_m \delta_{ij} - \frac{3}{2} \Phi_{ik,2} n_k n_j \right. \\ &\quad \left. - \frac{3}{2} \Phi_{jk,2} n_k n_i \right] f \left[\frac{\ell_t}{x_n} \right] &= \text{(wall, rapid part)} \\ +\nu \frac{\partial^2 \overline{v'_i v'_j}}{\partial x_k \partial x_k} &= \text{(viscous diffusion)} \\ +\frac{\partial}{\partial x_k} \left[c_k \overline{v'_k v'_m} \frac{k}{\varepsilon} \frac{\partial \overline{v'_i v'_j}}{\partial x_m} \right] &= \text{(turbulent diffusion)} \\ -g_i \beta \overline{v'_j \theta'} - g_j \beta \overline{v'_i \theta'} &= \text{(buoyancy production)} \\ -\frac{2}{3} \varepsilon \delta_{ij} &= \text{(dissipation)}\end{aligned}$$

► The exact transport equation for turbulent heat flux vector $\overline{v'_i \theta'}$ reads

$$\begin{aligned} \frac{\partial \overline{v'_i \theta'}}{\partial t} + \frac{\partial}{\partial x_k} \overline{v_k v'_i \theta'} &= \underbrace{-\overline{v'_i v'_k} \frac{\partial \bar{\theta}}{\partial x_k}}_{P_{i\theta}} - \underbrace{v'_k \theta' \frac{\partial \bar{v}_i}{\partial x_k}}_{\Pi_{i\theta}} - \underbrace{\frac{\theta'}{\rho} \frac{\partial p'}{\partial x_i}}_{D_{i\theta,t}} - \underbrace{\frac{\partial}{\partial x_k} \overline{v'_k v'_i \theta'}}_{D_{i\theta,t}} \\ + \alpha \underbrace{\frac{\partial}{\partial x_k} \left(v'_i \frac{\partial \theta'}{\partial x_k} \right)}_{D_{i\theta,\nu}} + \nu \underbrace{\frac{\partial}{\partial x_k} \left(\theta' \frac{\partial v'_i}{\partial x_k} \right)}_{\varepsilon_{i\theta}} - (\nu + \alpha) \underbrace{\frac{\partial v'_i}{\partial x_k} \frac{\partial \theta'}{\partial x_k}}_{\varepsilon_{i\theta}} - \underbrace{g_i \beta \overline{\theta'^2}}_{G_{i\theta}} \end{aligned}$$

► The exact k equation reads

$$\frac{\partial k}{\partial t} + \frac{\partial \bar{v}_j k}{\partial x_j} = -\overline{v'_i v'_j} \frac{\partial \bar{v}_i}{\partial x_j} - \frac{\partial}{\partial x_j} \left[\frac{1}{\rho} \overline{v'_j p'} + \frac{1}{2} \overline{v'_j v'_i v'_i} - \nu \frac{\partial k}{\partial x_j} \right] - \nu \frac{\partial v'_i}{\partial x_j} \frac{\partial v'_i}{\partial x_j}$$

► The exact $\overline{v'_i v'_j}$ equation reads

$$\begin{aligned} \frac{\partial \overline{v'_i v'_j}}{\partial t} + \frac{\partial}{\partial x_k} (\bar{v}_k \overline{v'_i v'_j}) &= -\overline{v'_j v'_k} \frac{\partial \bar{v}_i}{\partial x_k} - \overline{v'_i v'_k} \frac{\partial \bar{v}_j}{\partial x_k} \\ - \frac{\partial}{\partial x_k} \left(\overline{v'_i v'_j v'_k} + \frac{1}{\rho} \delta_{jk} \overline{v'_i p'} + \frac{1}{\rho} \delta_{ik} \overline{v'_j p'} - \nu \frac{\partial \overline{v'_i v'_j}}{\partial x_k} \right) \\ + \frac{1}{\rho} p' \left(\frac{\partial v'_i}{\partial x_j} + \frac{\partial v'_j}{\partial x_i} \right) &- g_i \beta \overline{v'_j \theta'} - g_j \beta \overline{v'_i \theta'} - 2\nu \frac{\partial v'_i}{\partial x_k} \frac{\partial v'_j}{\partial x_k} \end{aligned}$$

► The modelled k and ε equations

$$\begin{aligned} \frac{\partial k}{\partial t} + \bar{v}_j \frac{\partial k}{\partial x_j} &= \nu_t \left(\frac{\partial \bar{v}_i}{\partial x_j} + \frac{\partial \bar{v}_j}{\partial x_i} \right) \frac{\partial \bar{v}_i}{\partial x_j} + g_i \beta \frac{\nu_t}{\sigma_\theta} \frac{\partial \bar{\theta}}{\partial x_i} \\ - \varepsilon + \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_t}{\sigma_k} \right) \frac{\partial k}{\partial x_j} \right] \\ \frac{\partial \varepsilon}{\partial t} + \bar{v}_j \frac{\partial \varepsilon}{\partial x_j} &= \frac{\varepsilon}{k} c_{\varepsilon 1} \nu_t \left(\frac{\partial \bar{v}_i}{\partial x_j} + \frac{\partial \bar{v}_j}{\partial x_i} \right) \frac{\partial \bar{v}_i}{\partial x_j} \\ + c_{\varepsilon 1} g_i \frac{\varepsilon}{k} \frac{\nu_t}{\sigma_\theta} \frac{\partial \bar{\theta}}{\partial x_i} - c_{\varepsilon 2} \frac{\varepsilon^2}{k} + \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\nu_t}{\sigma_\varepsilon} \right) \frac{\partial \varepsilon}{\partial x_j} \right] \end{aligned}$$