

CFD WITH OPENSOURCE SOFTWARE

A COURSE AT CHALMERS UNIVERSITY OF TECHNOLOGY
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Project description:

A pimpleFoam tutorial for channel flow, with respect to different LES models.

Developed for OpenFOAM-1.6-ext
Requires: FreeCAD, enGrid, pyFoam

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Disclaimer: This is a student project work, done as part of a course where OpenFOAM and some other OpenSource software are introduced to the students. Any reader should be aware that it might not be free of errors. Still, it might be useful for someone who would like learn some details similar to the ones presented in the report and in the accompanying files.

November 4, 2011

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Chapter 1

Introduction

This report will describe how to pre-process, process and post-process a case based on the incompressible Navier-Stokes equations together with approximations based on different LES-models (Large-Eddy Simulation). No validation is performed, since it is just a fictive case. The intention is that the reader should be able to follow the description of the case like a tutorial.

The case is chosen to resemble a generic vortex meter. The principle of the vortex meter is that vortices are developing downstream a bluff body (in this case represented by a slender rod). The vortex shedding frequency is proportional to the flow rate. The aim of the case description/tutorial is to compare a two different LES models on this simple structure, defined by a squared channel with unchanged boundary conditions at the inlet. The comparison will be based on samples of the pressure fluctuations downstream a slender rod placed in the center of the channel.

Furthermore the project also aims to give some background to and clarify differences between different turbulence models. This work will be performed on a more theoretical level.

Chapter 2

Tutorial

2.1 Case definition

The structure on which the simulation will be applied is a very coarse and simplified model of a vortex flowmeter, see Figure 2.1. It is built up of a channel with a cross sectional area of 200x200 mm. The channel length is set to 1000 mm. The slender rod is of equal height as the channel. It is 20x20 mm wide and placed in the center of the channel, with the upstream face positioned 200 mm from the inlet.

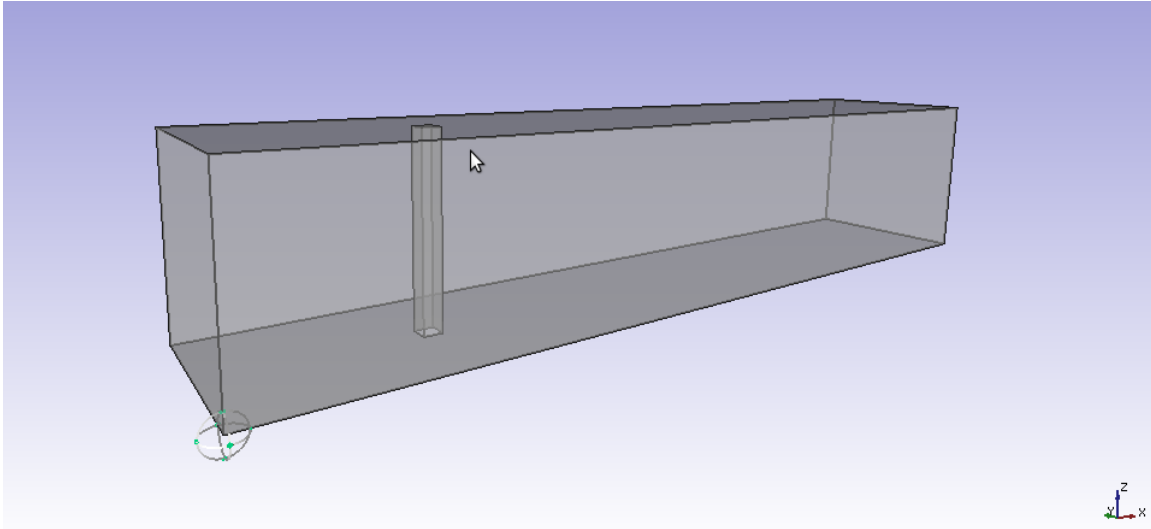


Figure 2.1: Typical CAD drawing which meshing was first intended to be based upon

The fluid of interest is water with a kinematic viscosity, ν , of $1.0 \times 10^{-6} \text{ m}^2/\text{s}$. The inlet velocity, v , is set to 0.25 m/s which would yield a vortex shedding frequency, F , of 2.5 Hz according to Equation (2.1). That would yield a simulation time of at least 10 seconds to generate a well developed flow and distinct vortices. Data need to be generated at around 25 Hz to avoid undersampling.

$$F = \frac{Sr \cdot v}{L} \quad (2.1)$$

where F = Shedding frequency, Sr = Strouhal number = 0.2, v = velocity = 1 m/s, L = bluff body diameter = 0.04 m. The Reynolds number is defined according to Equation (2.2)

$$Re = \frac{vL}{\nu} \quad (2.2)$$

where L equals the hydraulic diameter. For a pipe, that would correspond to the internal diameter. In this case, a channel with a squared cross sectional area is simulated which is why we need to define L as in Equation (2.3)

$$L = \frac{4A}{P} \quad (2.3)$$

where A indicates cross sectional area, and P the wetted perimeter. For the given dimensions, the resulting Reynolds number is evaluated to $Re = 50000$ which is well inside the turbulent region. Hence applying LES to such a problem would be appropriate.

2.2 Preprocessing

In the preprocessing step, creating the mesh causes some problems. The work has been divided into two branches, a non working one, and one with better results. Nevertheless, both versions are described below.

2.2.1 Meshing - Non working solution

Initially the case structure was intended to be constructed with an open source CAD software. FreeCAD seemed to be a fairly simple alternative. It is available as an Open Source CAD alternative based on OpenCasCade, Qt and Python. There are possibilities to create meshes in STL format, which makes it possible to export the meshes to enGrid (a meshing software with OpenFoam interface). Generating the structure is done by adding two cuboids of appropriate sizes to the drawing. The larger one representing the channel, the smaller representing the rod. The two objects are then joined in a boolean operation by taking the difference. Meshing suitable for OpenFoam is however difficult in FreeCAD. As mentioned, it is possible to generate a basic mesh, which can be exported as a *.stl file. If the reader is interested in how the structure is created in FreeCAD, a short list of important steps will follow.

1. Download and install FreeCAD which is available from Ubuntu Software Center.
2. Start FreeCAD and create a new empty document. (Ctrl+N)
3. Right click the drawing and select **Navigation Style > Inventor Navigation**.
(It makes the mouse behave as in ParaView for example.)
4. Add two boxes to the drawing.
5. Highlight **Box** in the Project Navigator.
6. Optional: Press **Ctrl+D** and set transparency of **Box** to 50 %.
7. Change the size of **Box** to h=200 mm, l=1000 mm, w = 200 mm.
8. Highlight **Box001** in the Project Navigator.
9. Change the size of **Box001** to h=200 mm, l=20 mm, w = 20 mm.

10. Change the position of **Box001** to x=200 mm, y=90 mm, z=0 mm.
11. Press **Shift** and left click both boxes in the Project Navigator window.
12. Navigate to **Part > Boolean** in the menu and select it.
13. Select **Difference** as boolean operation and make sure **Box001** is subtracted from **Box**.
14. Click **Apply**.
15. Click **Meshes > Create mesh from geometry...**
16. Press **Ctrl+E** and export the mesh by naming it with an *.stl extension and selecting **Mesh formats** from the dropdown list before saving.

Having a *.stl file, the open source mesh generator enGrid is chosen as an interface between FreeCAD and OpenFoam. enGrid is a rather new software, developed by Oliver Gloth in 2008. The meshing in enGrid is based on the Netgen libraries, which results in unstructured tetrahedral meshes with possibilities of prismatic boundary layers. The installation of enGrid under Ubuntu 11.04 is not as simple as for FreeCAD. A bit of interaction from the user is needed. To avoid introducing any errors while installing enGrid of the reader is hereby advised to visit enGits webpage for reliable instructions for different operating systems.

1. Navigate to <http://engits.eu/wiki/index.php/Installation>.
2. Follow the instructions according to your OS.
3. Remark: If enGrid is installed in your home directory, adding

```
alias enGrid='$HOME/engrid/run.bash'
```

to your `.bashrc` file enables you to start the program by just typing enGrid in your terminal.

The aim of using enGrid is to get a proper set of files for OpenFoam. To start with, the earlier created *.stl file is imported to enGrid. When meshing in enGrid, you first need to set boundary codes to the different faces. In this case the structure consists of 10 faces, 8 of them are defined as walls while the other are defined as inlet and outlet patches. One also need to add a volume and point out how the normal vectors of the faces are pointed, i.e on which side of the face the interesting volume is located. That is done by coloring the boundary faces green or yellow (strange). Next, meshing of the boundary faces is performed. If needed, a prismatic boundary layer can be generated afterwards.

When the surface mesh is generated, the next step is to create the volume mesh. Here is where I think enGrid has it weakness. Perhaps it is possible to create a mesh of suitable density but the documentation is rather sparse so it was difficult to figure out how. It seems as if the mesh resolution of the boundary faces completely determines the size of the internal cells and it was difficult to create small enough cells throughout the volume without running into memory problems on an 8 GB machine. A coarse enough mesh distribution to avoid filling up the memory could look something like the one illustrated in Figure 2.2. Unfortunately it is not enough for a reasonable solution to this case.

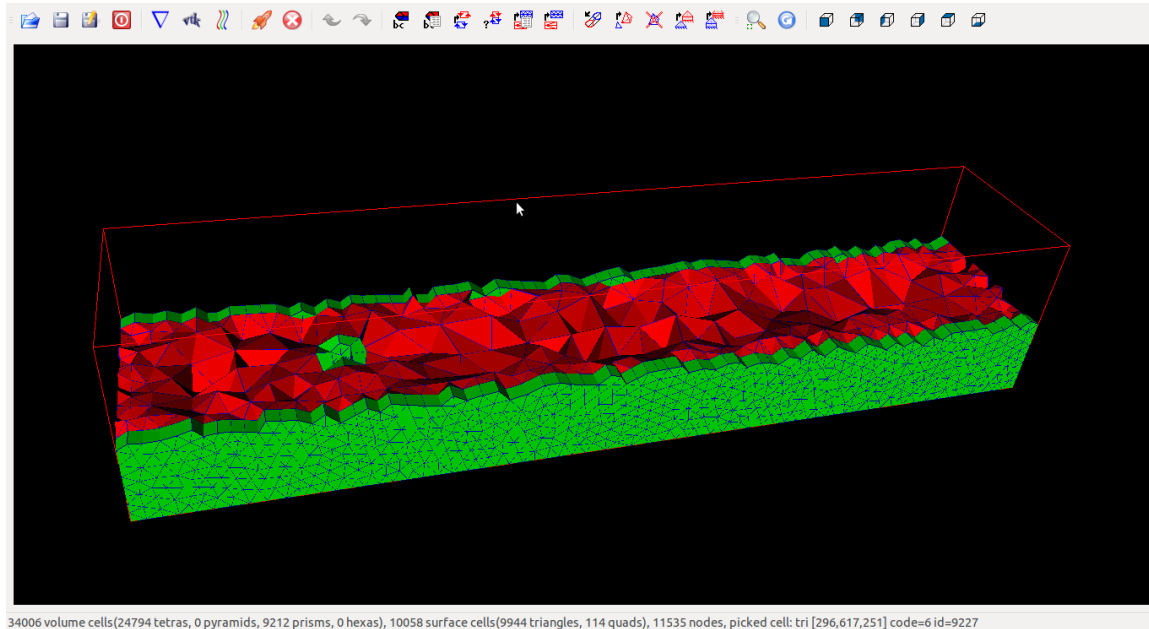


Figure 2.2: Unable to generate a grid fine enough throughout the volume.

If a solution to the volume meshing is found, the nice thing about enGrid is that it has an interface to OpenFoam. If everything is set up correctly one can export the grid separately, or the grid in combination with solver parameters. It is however a limited number of solvers that are supported, currently simpleFoam and rhoSimpleFoam from version 1.5 to version 1.6.x. Below, enumerated step by step instructions for enGrid is listed, to achieve the mesh in Figure 2.2

1. Start enGrid (by typing enGrid in terminal if you followed the remark)
2. Click **Import** > **STL** and select the *.stl file you previously created
3. Rotate the drawing so you get a decent view of the structure.
4. Hover over the structure with the mouse and select a face by pressing **p**
5. Press **s** to set the boundary code for that face. The Feature Angle can be left at 45.00 but make sure the Boundary Code is incremented for each new boundary.
6. Press **e** to edit boundary codes. For this case it is enough if you have ended up with 10 boundaries.
7. Set the BC-names to "wall1"- "rod4", "rod1"- "rod4" , "inlet" and "outlet"
8. Set the BC-types of the wall and rod faces to "wall" and to "patch" for the inlet and outlet
9. Press **v** to select which boundaries should be visible.

10. Now it's time to generate the surface mesh. Click on **Mesh > edit surface parameters**
11. Mark all faces and insert the following code under "rules for mesh resolution".

```
inlet <OR>
outlet <OR>
wall1 <OR>
wall2 <OR>
wall3 <OR>
wall4 <OR>
rod1 <OR>
rod2 <OR>
rod3 <OR>
rod4 = 2;
```

(Remark: You need to change the number if you want another resolution.)

12. Click **Mesh > create/improve surface mesh**
13. Press **e** to edit boundary conditions
14. Add a new volume named "vol"
15. In the new column, select "green", which tells the solver that the control volume is located inside the boundaries.
16. Now, click **Mesh > create/improve volume mesh (NETGEN)**
17. Click on "vol" in the dialog and then press **OK**
18. If needed, a prismatic boundary layer can be generated under **Mesh > create prismatic boundary layer**
19. The mesh can be examined as in Figure 2.2 by making appropriate settings under **Display options**.

Exporting the mesh to OpenFoam is also possible from enGrid, but since the mesh generated in this tutorial isn't fine enough, that functionality has not been completely investigated. It is also important to point out that the exporting feature is only available for a limited number of solvers. One might have to do some additional changes by hand to make use of the files in an arbitrary case.

2.2.2 Meshing - Working solution

Since it is difficult to create a mesh with a reasonable cell size ratio in enGrid, it is abandoned and a structured mesh is created manually. The case files can be found in the homepage of the MSc/PhD course in CFD with OpenSource software¹, held at Chalmers in 2011. The `blockMeshDict` file which specifies the location of the vertices, faces and blocks should be placed in `$CASE_DIR/constant/polyMesh/blockMeshDict`. (From here on, the directory of the case will be referred to as `$CASE_DIR`.) Creating a `blockMeshDict` that describes the block distribution shown in Figure 2.3 is enough. Figure 2.3 illustrates a very simple extruded block distribution around a rod (slender dark block in the center). Running `blockMesh` on such a `blockMeshDict` file results in

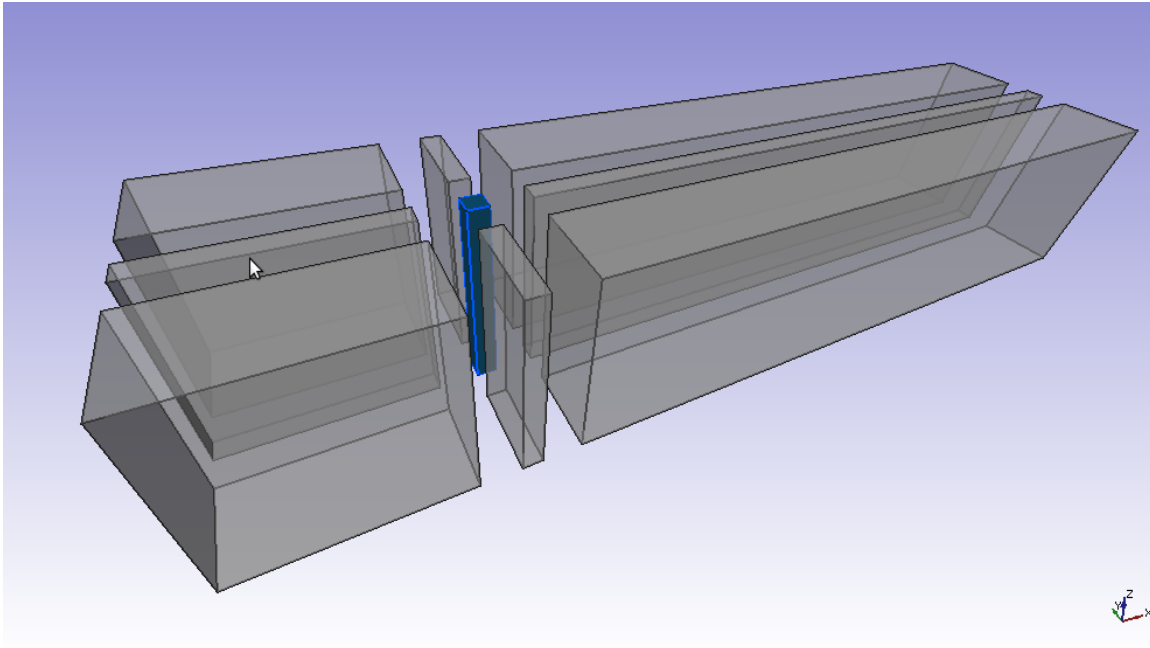


Figure 2.3: Suggested block distribution (not exactly how it was made in this project)

a structured mesh with equal sized cells of 1 cm^3 if everything is set up correctly. That is a very coarse grid, which makes the solutions unreliable, but the aim of this project is just to learn the basic procedures in solving a case. However, to resolve the regions around the rod, where generation of vortices was anticipated, a finer mesh is needed. Hence a utility in OpenFoam, called `refineMesh` is described below.

¹http://www.tfd.chalmers.se/~hani/kurser/OS_CFD


```

}

patchLocalCoeffs
{
    patch outside; // Normal direction is facenormal of zero'th face of patch
    tan1 (1 0 0);
}

// List of directions to refine
directions
(
    tan1
    tan2
    tan3
//normal
);

// Whether to use hex topology. This will
// - if patchLocal: all cells on selected patch should be hex
// - split all hexes in 2x2x2 through the middle of edges.
useHexTopology true;

// Cut purely geometric (will cut hexes through vertices) or take topology
// into account. Incompatible with useHexTopology
geometricCut false;

// Write meshes from intermediate steps
writeMesh false;

// ***** //

```

Here, we see that `refineMeshDict` makes use of the previously defined cell set `c0`. The utility is executed by typing `refineMesh -dict -overwrite` in `$CASE_DIR`. As seen in the comments of `refineMeshDict`, the utility will divide the cells of the defined region in 2x2x2 parts. These actions will result in a finer grid near the rod, see Figure 2.4. (Screen resolution might affect the appearance.)

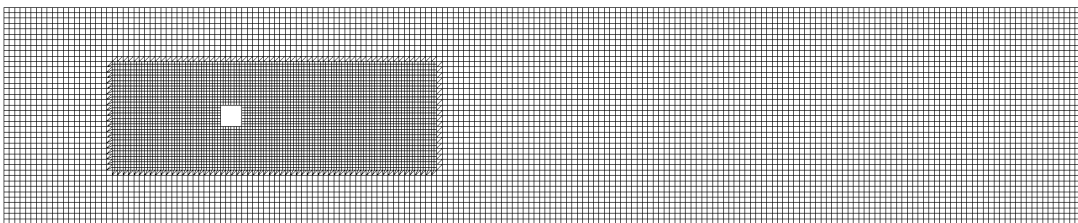


Figure 2.4: Result from `refineMesh` on mesh specified in the `blockMeshDict` file

2.2.3 Setting up the boundary conditions

The boundary conditions are defined in files located in `$CASE_DIR/0` named `U`, `epsilon`, `k`, `nuSgs`, `nuTilda`, `nut` and `p`. The content of these files will not be presented in detail here.

2.2.4 Setting up the solver

To set up information for the solver there is a need to do some changes to the files located in the `$CASE_DIR/system` directory. To start with, `$CASE_DIR/system/controlDict` is specified as follows.

```
// * * * * * //

application    pimpleFoam;

startFrom      latestTime;

startTime      0;

stopAt         endTime;

endTime        15;

deltaT         0.005;

writeControl   adjustableRunTime;

writeInterval  0.04;

purgeWrite     0;

writeFormat    ascii;

writePrecision 6;

writeCompression uncompressed;

timeFormat     general;

timePrecision  6;

runTimeModifiable true;

adjustTimeStep yes;

maxCo          1;

functions
{
    probes
    {
        // Where to load it from
        functionObjectLibs ( "libsampling.so" );

        type                probes;

        // Name of the directory for probe data
        name                 probes;

        // Write at same frequency as fields
    }
}
```

```

outputControl    outputTime;
outputInterval  1;

// Fields to be probed
fields
(
    p U
);

probeLocations
(
    ( 0.01 0.1 0.1 )    // at inlet
    ( 0.19 0.1 0.1 )   // at front face of rod
    ( 0.23 0.1 0.1 )   // at back face of rod
    ( 0.99 0.1 0.1 )   // at outlet
);
}
}

// ***** //

```

As seen, `pimpleFoam`² is selected as solver, with an adjustable timestep. The timestep is adjusted so the Courant number doesn't exceed 1. Some probes are added for pressure and velocity for post-processing. Probe number three with coordinates (0.23 0.1 0.1) is located right behind the rod and will later be used to compare the time history of the pressure for two different turbulence models.

Then, a closer look at the `$CASE_DIR/system/turbulenceProperties` is needed.

```

// * * * * * //

simulationType LESModel;

// ***** //

```

It is a rather short file, only specifying that a LES model will be used for turbulence modelling. This tells the solver to create a base class of type `LESModel`. The definition of `LESModel` is found in `$FOAM_SRC/turbulenceModels/incompressible/LES/LESModel`. That class "defines the basic interface for an incompressible flow SGS³ model, and encapsulates data of value to all possible models." according to it's header. If the `simulationType` is set to `LESModel` a `$CASE_DIR/system/LESProperties` is needed, where the LES model has to be defined. It consists of the following lines.

```

// * * * * * //

LESModel        Smagorinsky;

delta            cubeRootVol;

printCoeffs     on;

cubeRootVolCoeffs
{
    deltaCoeff    1;
}

```

²`pimpleFoam` will be discussed in Chapter 2.3

³SGS=Sub Grid Scale

```
PrandtlCoeffs
{
    delta          cubeRootVol;
    cubeRootVolCoeffs
    {
        deltaCoeff    1;
    }

    smoothCoeffs
    {
        delta          cubeRootVol;
        cubeRootVolCoeffs
        {
            deltaCoeff    1;
        }

        maxDeltaRatio  1.1;
    }

    Cdelta          0.158;
}

vanDriestCoeffs
{
    delta          cubeRootVol;
    cubeRootVolCoeffs
    {
        deltaCoeff    1;
    }

    smoothCoeffs
    {
        delta          cubeRootVol;
        cubeRootVolCoeffs
        {
            deltaCoeff    1;
        }

        maxDeltaRatio  1.1;
    }

    Aplus          26;
    Cdelta          0.158;
}

smoothCoeffs
{
    delta          cubeRootVol;
    cubeRootVolCoeffs
    {
        deltaCoeff    1;
    }
}
```

```

    maxDeltaRatio    1.1;
}

```

```
// ***** //
```

Here, the selection of LES model is made by setting the `LESModel` variable. For this project, two models have been compared, so the variable has been set to `Smagorinsky` in one of the cases and to `oneEqEddy` in the other. These models are defined in the `$FOAM_SRC/turbulenceModels/incompressible/LES/` directory. The models are further explained in Chapter 3

2.3 Processing the case with `pimpleFoam`

Referring to the exact definition⁴ of `pimpleFoam` it is a large time-step transient solver for incompressible flow using the PIMPLE (merged PISO-SIMPLE) algorithm. PISO is an acronym for Pressure Implicit Splitting of Operators for time dependent flows while SIMPLE stands for Semi-Implicit Method for Pressure Linked Equations which is used for steady state problems⁵. In the SIMPLE algorithm a pressure correction term is used while the velocity corrections are neglected because they are unknown. This results in rather slow convergence. The PISO algorithm also neglects the velocity correction in the first step, but then performs one in a later stage, which leads to additional corrections for the pressure⁶.

The simulation is run by simply typing `pyFoamPlotRunner.py pimpleFoam >& log` in the `$CASE_DIR`. `pyFoamPlotRunner.py` is part of the `pyFoam` package. If it has been set up correctly, it will generate residual plots while the simulation is performed. These plots will not be presented in this report. Judging from the logfiles, the `oneEqEddy` model takes longer time than `Smagorinsky` to execute. Comparing the execution times at a simulation time of 10 s for example, `Smagorinsky` has run for 7146.36 s while `oneEqEddy` shows a value of 8740.53 s.

⁴<http://www.openfoam.com/features/standard-solvers.php>

⁵<http://projects.exeter.ac.uk/fluidflow/Courses/ComputationalEng3213-4/slidesCFD/lecture3-slides.pdf>

⁶Joel H. Ferziger / Milovan Peric, *Computational Methods for Fluid Dynamics*, 3rd Edition, 2002 Springer

2.4 Post-processing

As earlier described in the `controlDict` file, probes have been set up to sample the pressure at well defined positions. It is interesting to look at data gathered with the probe located at position $(0.23 \ 0.1 \ 0.1)$ which would be located approximately at the position of the pressure sensor of a vortex meter. The sampled signal is plotted vs time. This is done for two LES models (Smagorinsky and oneEqEddy) to show possible differences between the solutions. The pressure samples are located in `$CASE_DIR/probes/0/p`. Using `gnuplot`, Figure 2.5 is created which shows a difference in pressure readings from the two simulations.

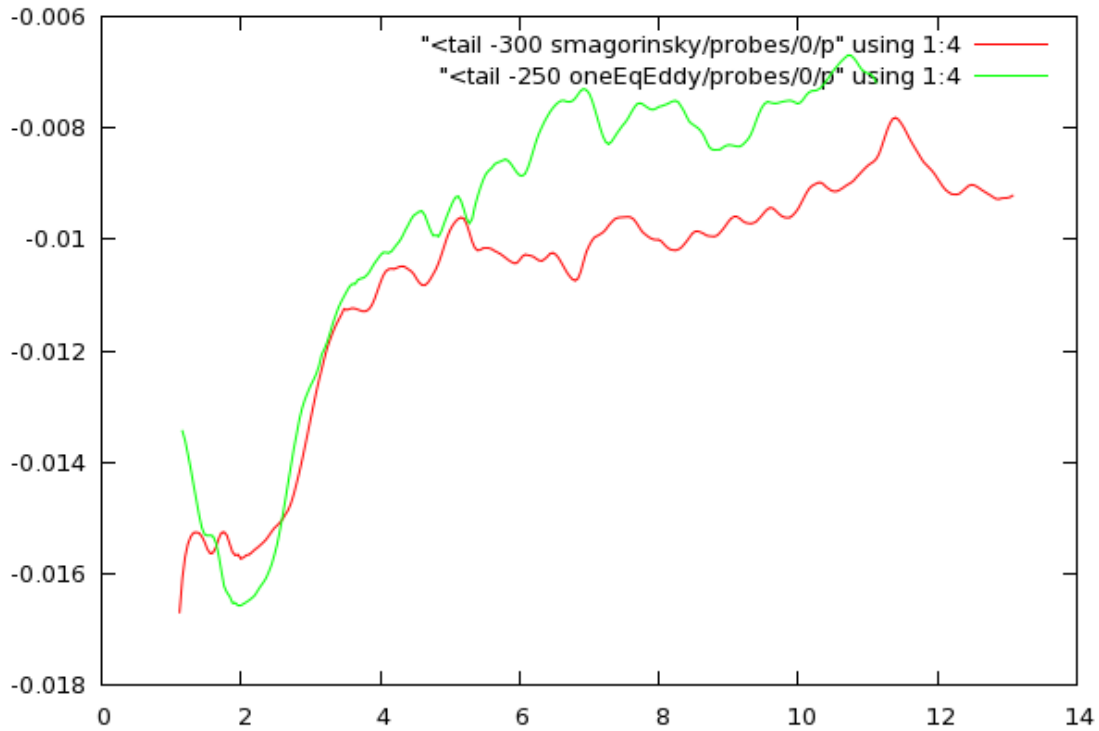


Figure 2.5: Plot of pressure on the back face of the rod for two different LES models

At a later stage, one might be interested in plotting it vs frequency as well, but the dataset is probably too sparse to get any interesting information from these cases.

The pressure distribution can also be illustrated graphically in ParaView. In Figure 2.6 the pressure distribution after a simulation time of 8 s is illustrated.

smagorinsky

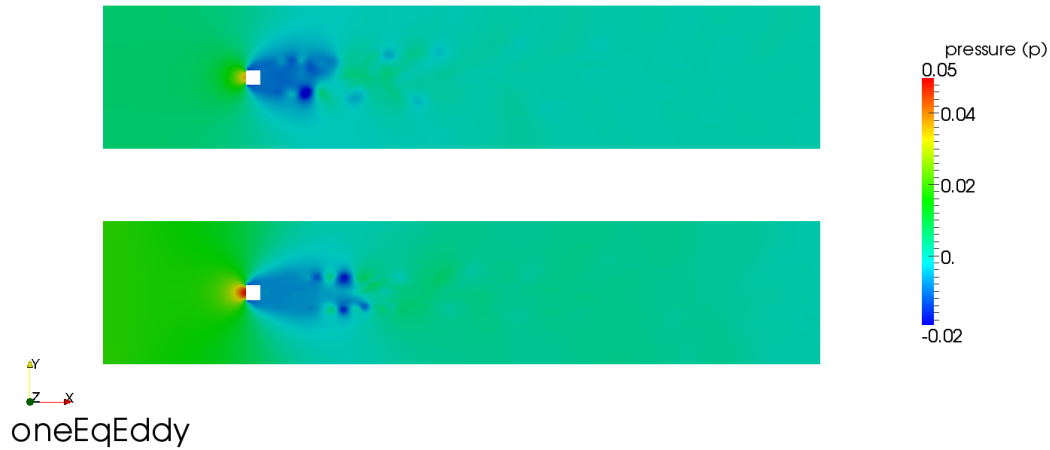


Figure 2.6: Pressure comparison at $t = 8s$, Smagorinsky vs oneEqEddy

The velocity distribution for the same time is illustrated in Figure 2.7. Here the two models clearly show different results.

smagorinsky

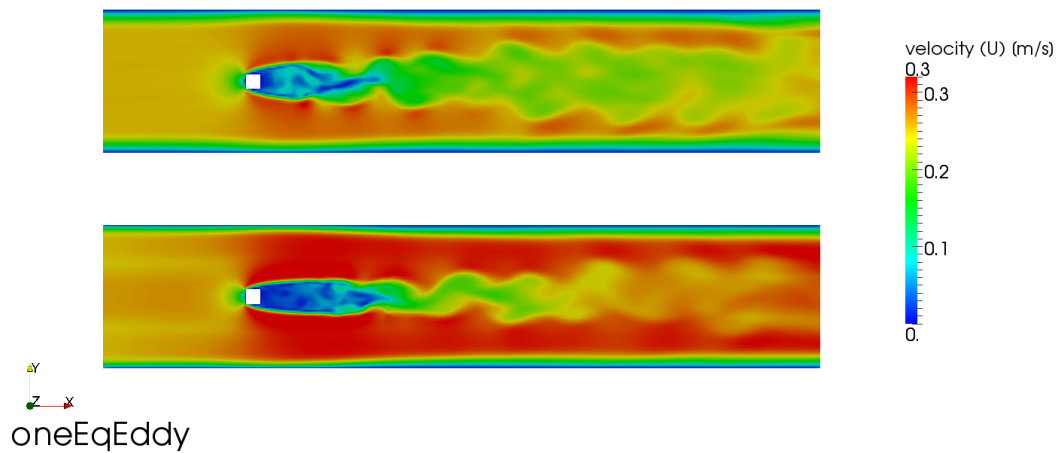


Figure 2.7: Velocity comparison at $t = 8s$, Smagorinsky vs oneEqEddy

2.5 Reduction of the near wall eddy viscosity

2.5.1 General discussion regarding the Smagorinsky model

According to Ferziger and Peric⁷ the Smagorinsky model tends to overestimate the eddy viscosity, μ_t , close to the surfaces of the channel. The eddy viscosity is one component of the effective viscosity, μ_{eff} , which is defined as:

$$\mu_{eff} = \mu + \mu_t \quad (2.4)$$

where μ , the other component, denotes the laminar dynamic viscosity. μ_t is defined as:

$$\mu_t = C_S^2 \rho \Delta^2 |\bar{S}| \quad (2.5)$$

where $C_S \simeq 0.2$ is a model parameter and assumed constant. ρ represents the fluid density while Δ equals the filter cutoff width, i.e. the characteristic length scale of the SGS eddies. \bar{S} represents the strain rate of the resolved field. Ferziger and Peric suggest that the eddy viscosity should be damped by using the model for van Driest damping, commonly used in RANS models:

$$C_S = C_{S0} \left(1 - e^{-n^+/A^+}\right)^2 \quad (2.6)$$

In OpenFoam however, the damping is derived by changing the filter width, depending on the distance from the wall. In `$FOAM_SRC/turbulenceModels/incompressible/LES/vanDriestDelta/vanDriestDelta.C` we see that delta is calculated according to:

$$\Delta = \min \left(\Delta_{mesh}, \left(\frac{\kappa}{C_\Delta} \right) y \left(1 - e^{-y^+/A^+}\right) \right) \quad (2.7)$$

where Δ_{mesh} is the cubic root of the cell volume, $\kappa = 0.4187$ is the von Kármán constant, $C_\Delta = 0.158$, $A^+ = 26$, y represents the distance to the wall, while y^+ describes the dimensionless distance to the wall, calculated from the wall shear stress⁸. This also reduces μ_t near the walls.

2.5.2 Applying vanDriest damping to the tutorial case

The eddy viscosity wasn't reduced in the main tutorial of this project since `delta` was set to `cubeRootVol` in `$CASE_DIR/constant/LESProperties` and might be one of the reasons why we see differences between the different LES models. Comparing the two models in Figure 2.7 we see that the fluid elements seem to interact more with the rod, which can be caused by the overestimated eddy viscosity. Interested readers can investigate the effects of a dampened eddy viscosity in the near wall region. If so, change `delta` to `vanDriest` in `$CASE_DIR/constant/LESProperties` instead.

⁷Joel H. Ferziger / Milovan Peric, Computational Methods for Fluid Dynamics, 3rd Edition, 2002 Springer

⁸<http://www.modlab.lv/publications/mep2008/pdfs/97-102.pdf>

Making that for a simplified mesh, the pressure fluctuations behind the backface of the rod show a small difference, as seen in Figure 2.8. The differences mainly consists of a slight offset in the signal. The dampened eddy viscosity doesn't seem to have any significant effect on the vortex shedding frequency. It is unclear how much the coarseness of the grid near the wall affects the vortex shedding as well. It could be that the differences in eddy viscosity would have a higher impact if a finer mesh was used. Notice that these pressures are sampled from a different case than

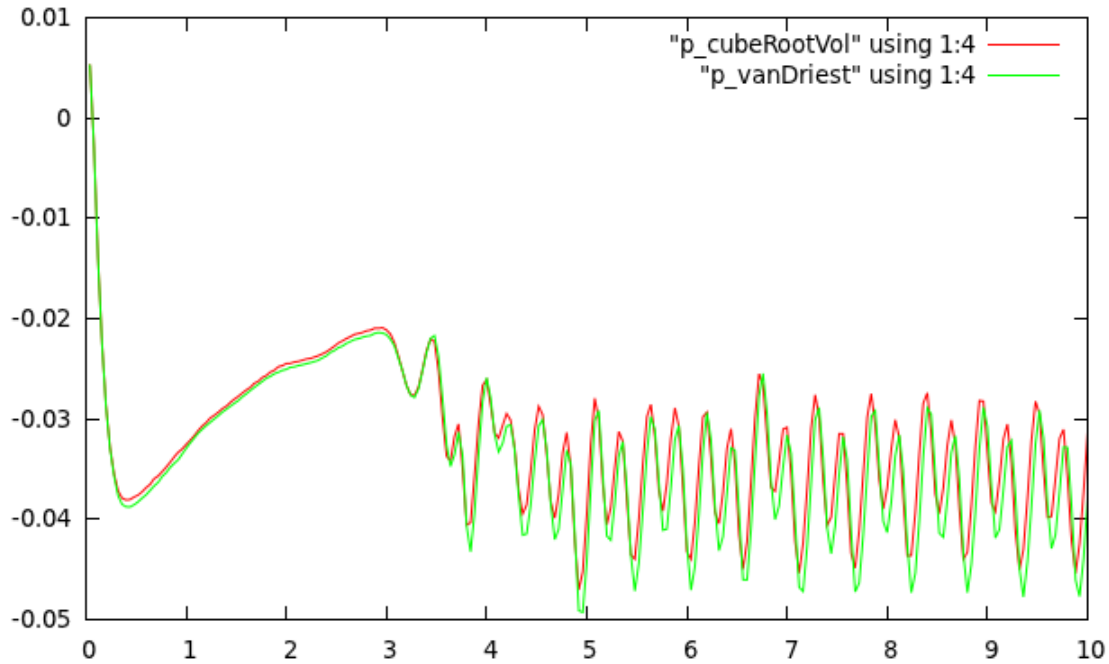


Figure 2.8: Pressure comparison for two different estimations of eddy viscosity.

those presented in Figure 2.6. The mesh used in this case has been flattened and the top and bottom faces are set to empty, which means that the model should not be influenced by any contribution from these boundaries. It might also be a bad idea to use LES on a 2D mesh since it is intended for 3D use. Another alternative than using the vanDriest damping is to implement a dynamic model instead.

Chapter 3

Investigation of LES models

Looking at the OpenFoam documentation there are numerous of different LES models to choose from, see Table 3.1. The LES models in OpenFoam are divided into anisochoric and isochoric types, definitions which are not explained in detail in the OpenFoam documentation. Isochoric means that the volume is kept constant, for example in a pressure vessel which doesn't expand though temperature is changing¹. Anisochoric models would then describe the opposite, i.e. consider a ballon where the volume is changing as soon as pressure or temperature change. The channel in the tutorial could be thought of as a constant volume since it is not expanding. Since the flow is incompressible it is assumed that an isochoric model would be applicable to the case.

Isochoric LES turbulence models	
Smagorinsky	Smagorinsky model
Smagorinsky2	Smagorinsky model with 3-D filter
dynSmagorinsky	Dynamic Smagorinsky
homogenousDynSmagorinsky	Homogeneous dynamic Smagorinsky model
dynLagrangian	Lagrangian two equation eddy-viscosity model
scaleSimilarity	Scale similarity model
mixedSmagorinsky	Mixed Smagorinsky/scale similarity model
dynMixedSmagorinsky	Dynamic mixed Smagorinsky/scale similarity model
kOmegaSSTAS	k-w-SST scale adaptive simulation (SAS) model
oneEqEddy	k-equation eddy-viscosity model
dynOneEqEddy	Dynamic k-equation eddy-viscosity model
locDynOneEqEddy	Localised dynamic k-equation eddy-viscosity model
spectEddyVisc	Spectral eddy viscosity model
LRDDiffStress	LRR differential stress model
DeardorffDiffStress	Deardorff differential stress model
SpalartAllmaras	Spalart-Allmaras model
SpalartAllmarasDDES	Spalart-Allmaras delayed detached eddy simulation (DDES) model
SpalartAllmarasIDDES	Spalart-Allmaras improved DDES (IDDES) model

Table 3.1: Available isochoric LES models in OpenFoam

Judging from the results of the tutorial, the `Smagorinsky` and `oneEqEddy` models behave differently. Looking at them from a theoretical point of view the differences are revealed.

¹http://en.wikipedia.org/wiki/Isochoric_process/

3.1 Smagorinsky

The Smagorinsky model, invented in 1969, is one of the earliest sub grid scale (SGS) models. The SGS represents the spatial region below the resolved length scale. The sparse description of the Smagorinsky model in `$FOAM_SRC/turbulenceModels/compressible/LES/Smagorinsky/Smagorinsky.H` states that:

$$\begin{aligned} \mathbf{B} &= 2/3 * \mathbf{k} * \mathbf{I} - 2 * \text{nuSgs} * \text{dev}(\mathbf{D}) \\ \mathbf{Beff} &= 2/3 * \mathbf{k} * \mathbf{I} - 2 * \text{nuEff} * \text{dev}(\mathbf{D}) \end{aligned}$$

where

$$\begin{aligned} \mathbf{D} &= \text{symm}(\text{grad}(\mathbf{U})); \\ \mathbf{k} &= (2 * \text{ck} / \text{ce}) * \text{delta}^2 * |\mathbf{D}|^2 \\ \text{nuSgs} &= \text{ck} * \text{sqrt}(\mathbf{k}) * \text{delta} \\ \text{nuEff} &= \text{nuSgs} + \text{nu} \end{aligned}$$

According to this description, \mathbf{B} represents the local SGS stresses, which is important on a SGS level. In Chapter 3.8.2 of *An Introduction to Computational Fluid Dynamics*² it is stated that Smagorinsky assumed the local SGS stresses R_{ij} to be proportional to the local rate of strain of the resolved flow $\bar{S}_{ij} = \frac{1}{2} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right)$. Hence, R_{ij} is defined in Equation (3.1) as:

$$R_{ij} = -2\mu_{SGS}\bar{S}_{ij} + \frac{1}{3}R_{ii}\delta_{ij} = -\mu_{SGS} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right) + \frac{1}{3}R_{ii}\delta_{ij} \quad (3.1)$$

The term $\frac{1}{3}R_{ii}\delta_{ij}$ ensures that the sum of the SGS stresses is equal to the kinetic energy of the SGS eddies. There exists a bit of confusion about the values of the modeling constants ck and ce according to a thread in www.cfd-online.com³. That is probably because the Smagorinsky model is implemented differently in OpenFoam compared to definition in literature. If \mathbf{B} and R_{ij} are compared, we see that they are similar but not identical. We see that the expressions differ by a factor of two. It can possibly be derived from a different definition of the strain rate tensor, $\bar{S}_{ij} = \frac{1}{2} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right)$, between literature and OpenFoam. Further, \mathbf{Beff} take the ordinary kinematic viscosity into account. Digging into the code of `Smagorinsky.C` a value of 0.094 is found for ck . ce equals 1.048 according to `$FOAM_SRC/turbulenceModels/compressible/LES/GenEddyVisc/GenEddyVisc.C`. According to literature, Equation (3.2) is an expression for the whole stress tensor which also includes the cross-stresses and Leonard stresses, described in Table 3.2.

$$\tau_{ij} = -2\mu_{SGS}\bar{S}_{ij} + \frac{1}{3}\tau_{ii}\delta_{ij} = -\mu_{SGS} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right) + \frac{1}{3}\tau_{ii}\delta_{ij} \quad (3.2)$$

Term	Definition	Interpretation
Leonard stresses	$L_{ij} = \rho \bar{u}_i \bar{u}_j - \rho \bar{u}_i \bar{u}_j$	Effects at resolved scale
Cross-stresses	$C_{ij} = \rho \bar{u}_i u'_j + \rho u'_i \bar{u}_j$	Interactions between SGS eddies and resolved flow
LES Reynolds stresses	$R_{ij} = \rho u'_i u'_j$	Interactions between SGS eddies

Table 3.2: SGS stress contributions

It is rather unclear if the introduction of the laminar kinetic viscosity in the expression for \mathbf{Beff} makes it identical to Equation (3.2).

²An Introduction to Computational Fluid Dynamics, The Finite Volume Method, 2nd Edition, H. K. Versteeg, W. Malalasekera, 1995-2007, Pearson Education Limited

³<http://www.cfd-online.com/Forums/openfoam-solving/65838-smagorinsky-model-details.html>

3.2 oneEqEddy

The oneEqEddy model on the other hand, uses a modeled balance equation for the SGS turbulent energy to simulate the behaviour of k . Looking in the `$FOAM_SRC/turbulenceModels/compressible/LES/oneEqEddy/oneEqEddy.H` file, the following explanation is found:

$$\begin{aligned} & \frac{d}{dt}(k) + \text{div}(U*k) - \text{div}(\text{nuEff}*grad(k)) \\ & = \\ & -B*L - c_e*k^{3/2}/\delta \end{aligned}$$

and

$$\begin{aligned} B &= 2/3*k*I - 2*\text{nuSgs}*dev(D) \\ B_{eff} &= 2/3*k*I - 2*\text{nuEff}*dev(D) \end{aligned}$$

where

$$\begin{aligned} D &= \text{symm}(grad(U)); \\ \text{nuSgs} &= c_k*\text{sqrt}(k)*\delta \\ \text{nuEff} &= \text{nuSgs} + \text{nu} \end{aligned}$$

The eddy viscosity seems to be modelled similarly as in the Smagorinsky model, but here an additional transport equation for the SGS turbulent kinetic energy, Equation (3.3), is solved as well.

$$\frac{\partial}{\partial t}(k) + \frac{\partial}{\partial x_i}(U_i k) - \frac{\partial}{\partial x_i} \left(\nu_{Eff} \frac{\partial}{\partial x_i}(k) \right) = -BL - \frac{c_e k^{3/2}}{\Delta} \quad (3.3)$$

Here, the first term on the left side describes the change of the turbulent SGS kinetic energy with respect to time. The second term describes convection and the third, diffusion. On the right hand side, two different source terms are listed. The first one $-BL$, is left without further description in the `oneEqEddy.H` file. Eugene de Villiers explains the background of the oneEqEddy model in his PhD thesis from 2006⁴. According to his thesis, the right hand side includes a term which he calls $\tau : \bar{S}$, which "represents the decay of turbulence from the resolved scales to the sub grid scales via the energy cascade". Looking into `$FOAM_SRC/turbulenceModels/compressible/LES/oneEqEddy/oneEqEddy.C` the member function `oneEqEddy::correct(const tmp<volTensorField>& gradU)` reveals a difference between the comments in `oneEqEddy.H` and the implementation in `oneEqEddy.C`.

```
void oneEqEddy::correct(const tmp<volTensorField>& gradU)
{
    GenEddyVisc::correct(gradU);

    tmp<volScalarField> G = 2.0*nuSgs_*magSqr(symm(gradU));

    tmp<fvScalarMatrix> kEqn
    (
        fvm::ddt(k_)
        + fvm::div(phi(), k_)
        - fvm::laplacian(DkEff(), k_)
    ==
        G
        - fvm::Sp(ce_*sqrt(k_)/delta(), k_)
    );
}
```

⁴<http://powerlab.fsb.hr/ped/kturbo/OpenFOAM/docs/EugeneDeVilliersPhD2006.pdf>

```

kEqn().relax();
kEqn().solve();

bound(k_, kMin_);

updateSubGridScaleFields();
}

```

Here, $-BL$ is represented by G which is defined in Equation (3.4).

$$G = 2.0 * \nu_{SGS} |\bar{S}|^2 \quad (3.4)$$

This corresponds to the decay term described in Eugenes thesis. The last term on the right hand side $-\frac{c_\epsilon k^{3/2}}{\Delta}$, corresponds to turbulent dissipation.

3.3 kOmegaSSTSAS

Just to give an example of an alternative implementation of a LES model, kOmegaSSTSAS is also mentioned. It is a model, developed in a European founded project called DESider, between 2004 and 2007. This model is based on the kOmegaSST which mixes two turbulence models, the $k - \omega$ model in the near wall region, where vorticity is expected and the $k - \epsilon$ model further out⁵. It also satisfy the Bradshaw assumption that $\tau_{ij} - \tau_{ii}$ in a boundary layer is proportional to k . The difference between the kOmegaSSTSAS and kOmegaSST is that they behave differently in unstable conditions. kOmegaSSTSAS, just as kOmegaSST, behaves like a steady state model for stable conditions. When the conditions become unsteady, μ_t is decreased which make kOmegaSSTSAS behave more like a LES model⁶.

⁵<http://www.cfd-online.com/Forums/openfoam-solving/58056-komega-turbulence-model-wwwopenfoamwikinet.html>

⁶DESider-A European Effor on Hybrid RANS-LES Modelling, W. Haase, M. Braza, A.Revell, 2009 Springer-Verlag Berlin Heidelberg