

## Note on the Body Force Propeller implementation

### Theory

In FINE™/Marine, the momentum equations include a body-force term  $\mathbf{f}_b$ , used to model the effects of a propeller without modeling the real propeller. There are numerous approaches for calculating  $\mathbf{f}_b$ , including simple prescribed distributions, which recover the total thrust  $\mathbf{T}$  and torque  $\mathbf{Q}$ , to more sophisticated methods which use a propeller performance code in an interactive way with the RANS solver to capture propeller-hull interaction and to distribute  $\mathbf{f}_b$  according to the actual blade loading. For the latter, special interface should be developed to extract effective wake from RANS solution and to produce  $\mathbf{f}_b$  calculated by a propeller performance code. FINE™/Marine does, however, include a prescribed body force with axial and tangential components. The radial distribution of forces, with components  $f_{bx}$  (axial),  $f_{br}$  (radial)=0 and  $f_{b\theta}$  (tangential), is based on non-iterative calculation of Stern et al.<sup>(1)</sup>, the Hough and Ordway<sup>(2)</sup> circulation distribution with optimum type from Goldstein<sup>(3)</sup>, and without any loading at the root and tip:

$$f_{bx} = A_x r^* \sqrt{1-r^*}, \quad f_{b\theta} = A_\theta \frac{r^* \sqrt{1-r^*}}{r^* (1-r'_h) + r'_h}$$

They represent body forces per unit volume normalized by  $\rho U^2/L$  where U is the reference velocity, L is a reference length, and  $\rho$  the fluid density. Coefficients are expressed as:

$$\begin{aligned} r^* &= \frac{r' - r'_h}{1 - r'_h}, \quad r'_h = R_H/R_P, \quad r' = r/R_P \\ r &= \sqrt{(y - Y_{PC})^2 + (z - Z_{PC})^2} \\ A_x &= \frac{C_T}{\Delta} \frac{105}{16(4 + 3r'_h)(1 - r'_h)} \\ A_\theta &= \frac{K_Q}{\Delta J^2} \frac{105}{\pi(4 + 3r'_h)(1 - r'_h)} \\ J &= \frac{2\pi U}{\Omega D_P} = \frac{U}{n D_P}, \quad D_P = 2R_P, \quad n = \Omega/2\pi \\ C_T &= \frac{2T}{\rho U^2 \pi R_P^2}, \quad K_T = \frac{T}{\rho n^2 D_P^4}, \quad K_Q = \frac{Q}{\rho n^2 D_P^5} \end{aligned}$$

and where  $C_T$  and  $K_Q$  are the thrust and torque coefficients, J is the advance coefficient, n is the number of rotations per second (rps),  $\Omega$  is the rotation speed,  $R_P$  is the propeller radius,  $R_H$  is the hub radius,  $\Delta$  is the mean chord length projected into the x-z plane (or actuator disk thickness), and  $Y_{PC}$  and  $Z_{PC}$  define the propeller center coordinates. As derived, these forces are defined over an "actuator cylinder" with volume defined by  $R_P$ ,  $R_H$ , and  $\Delta$ .

Integration of the body forces over this analytical volume exactly recovers the prescribed thrust  $\mathbf{T}$  and torque  $\mathbf{Q}$ :

$$T = \rho L^2 U^2 \iiint_A f_{bx} dA$$

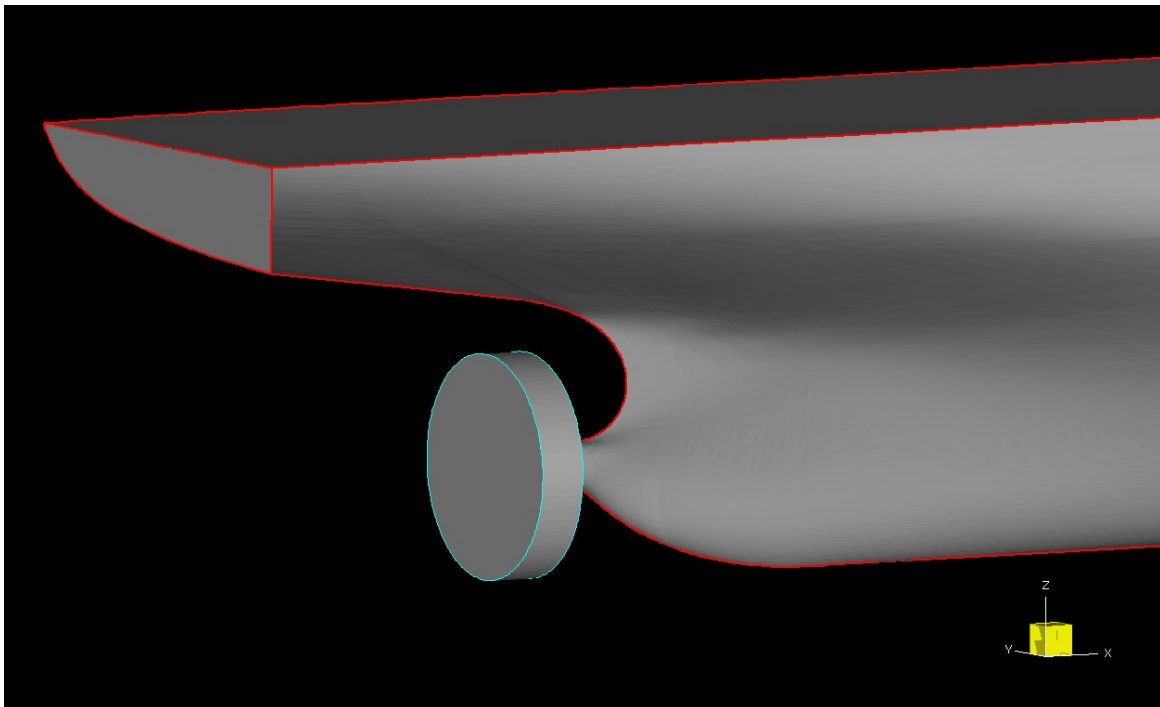
$$Q = \rho L^3 U^2 \iiint_A r f_{b\theta} dA$$

$$dA = 2\pi r \Delta r$$

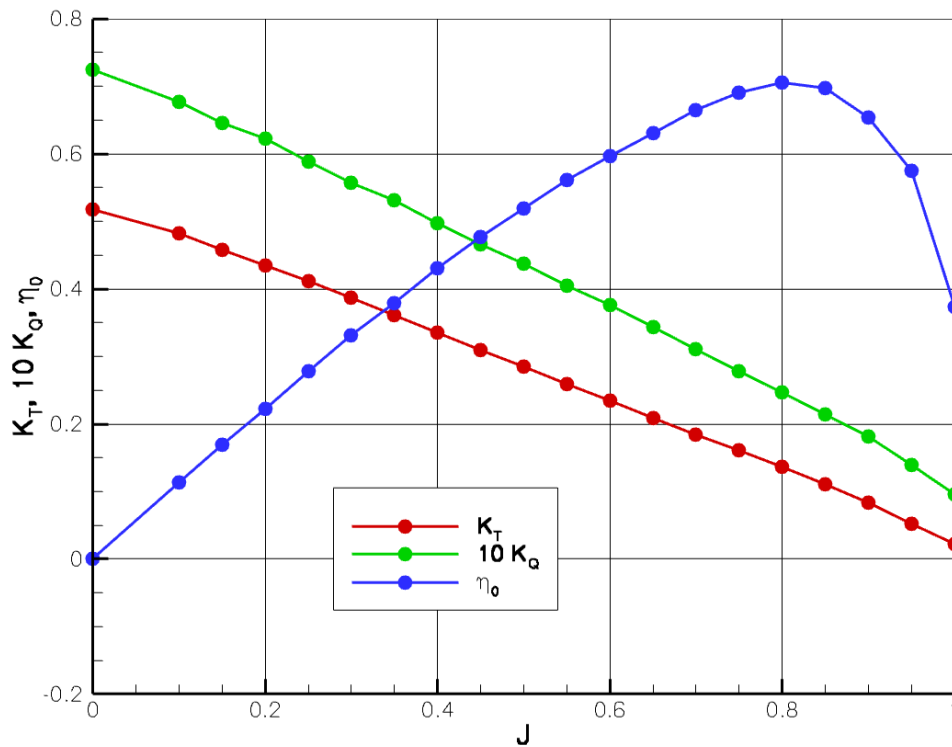
### Practice

In FINE™/Marine, user does not explicitly need to specify  $C_T$ ,  $K_Q$  and  $J$ , but prescribes the thrust  $\mathbf{T}$  and the torque  $\mathbf{Q}$ .  $A_x$  and  $A_\theta$  coefficients are then computed using the above relations.

As an example, a fictitious propeller region is materialized below where body force method must be activated:



If the propeller torque  $\mathbf{Q}$  is not known, it is however possible to estimate its magnitude from open water test of the propeller, if available. Such open water test usually provides the performance curve as illustrated for the propeller of the KRISO Container Ship (KCS) from the test case 2 of the Tokyo Workshop 2005:



For a prescribed thrust and/or an advance ratio parameter knowing the advance speed  $U$  and ratio  $J$  from the propeller rotation speed ( $n$  or  $\Omega$ ),  $K_Q$  is obtained from the performance curve and the propeller torque  $Q$  can be deduced from the propeller thrust  $T$  as:

$$Q = T D_P \frac{K_Q}{K_T}$$

As an example, for this case, the propeller condition was  $D_P=0.25\text{m}$  and  $J=0.732$ . From the performance curve, we have  $K_T=0.170$  and  $K_Q=0.029$  so the previous relation writes  $Q \sim 1.47 T$ .

## References

- (1) Stern, F., Kim, H.T., Patel, V.C., and Chen, H.C., "A Viscous-Flow Approach to the Computation of Propeller-Hull Interaction," *Journal of Ship Research*, Vol. 32, No. 4, December 1988, pp. 246-262.
- (2) Hough, G. and Ordway, D., "The Generalized Actuator Disk," Technical Report TAR-TR 6401, Therm Advanced Research, Inc., 1964.
- (3) Goldstein, S., "On the Vortex Theory of Screw Propellers", *Proc. of the Royal Society (A)* 123, 440, 1929.