

FVM for Fluid-Structure Interaction with Large Structural Displacements

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Objective

- Present a self-contained FSI application in a single software: **OpenFOAM**
- Examine FSI solution techniques, especially coupling algorithms
- Present advantages of single-software implementation and coupling

Topics

- Solution techniques in FSI simulations
 - Monolithic approach
 - Partitioned approach
- Partitioned approach with weak and strong coupling
 - Arbitrary Lagrangian-Eulerian (ALE) FVM solver for fluid flow
 - Updated Lagrangian FVM solver for elastic solid with large deformation
 - Iterative algorithm with adaptive under-relaxation for strong coupling
- OpenFOAM: Object-Oriented Computational Continuum Mechanics in C++
- Test cases: solid oscillations with large deformation and flow-induced vibration
- Summary and Future Work

Solution Techniques for Coupled Problems

- **Partitioned approach:** Picard Iterations
 - Optimal for weakly coupled FSI problems
 - Separate mathematical model for fluid and solid continua
 - Shared or different discretisation method: FVM and FEM
 - Coupling achieved by enforcing the kinematic and dynamic conditions on the fluid-solid interface
 - Strong coupling by additional iteration loop over partial solvers (need a convergence acceleration method)
- **Monolithic approach:** Simultaneous Solution
 - Appropriate when fluid-structure interaction is very strong
 - Good stability and convergence properties
 - In some cases may lead to ill-conditioned matrices or sub-optimal discretisation or solution procedure in fluid or solid region

Levels of Coupling

- Unified mathematical model: single equation set (Ivanković, UC Dublin)
- Unified discretisation method and boundary coupling consistency
- Unified solution procedure: fluid + structure matrix solved in a single solver

Partitioned Approach: Weak Coupling

- Solvers for fluid and solid are applied sequentially only once per time step
- Time lag between fluid and solid solution (explicit coupling)
- Special attention is needed in order to preserve 2nd order time accuracy

Artificial Added Mass Effect: Instability

- Occurs when weak coupling algorithm is used on FSI problems with incompressible fluid and a light/slender structure
- Numerical experiments show that the instability grows with
 - Decreasing time step size
 - Decreasing solid-fluid density ratio
 - Decreasing fluid viscosity
 - Decreasing solid stiffness

Selected FSI Simulation Approach

- Partitioned approach with weak and strong coupling algorithm
- ALE FVM solver for fluid with automatic mesh motion
- Updated Lagrangian FVM solver for elastic solid
- Adaptive under-relaxation and a strong coupling algorithm: Aitken

Mathematical Model

- Momentum equation for incompressible laminar flow in ALE formulation
- Continuity equation for incompressible flow in ALE formulation
- Space conservation law (SCL)
- Automatic mesh motion based on boundary deformation: Laplace equation

Discretisation

- Collocated 2nd order FVM method on a deforming mesh for the fluid flow
- 2nd order backward scheme for temporal discretisation
- 2nd order FEM for discretisation of automatic mesh motion

Solution Procedure

- Segregated solution procedure on a moving mesh
- PISO pressure-velocity coupling algorithm in fluid flows

Mathematical Model

- Incremental momentum equation in updated Lagrangian formulation

$$\int_{V_u} \rho_u \frac{\partial \delta \mathbf{v}}{\partial t} dV_u = \int_{S_u} \mathbf{n}_u \bullet \left(\delta \Sigma_u + \Sigma_u \bullet \delta \mathbf{F}_u^T + \delta \Sigma_u \bullet \delta \mathbf{F}_u^T \right) dS_u$$

- Incremental constitutive equation for St. Venant-Kirchhoff elastic solid

$$\delta \Sigma_u = 2\mu \delta \mathbf{E}_u + \lambda \text{tr}(\delta \mathbf{E}_u) \mathbf{I}$$

- Increment of Green-Lagrangian strain tensor

$$\delta \mathbf{E}_u = \frac{1}{2} \left[\nabla \delta \mathbf{u} + (\nabla \delta \mathbf{u})^T + \nabla \delta \mathbf{u} \bullet (\nabla \delta \mathbf{u})^T \right]$$

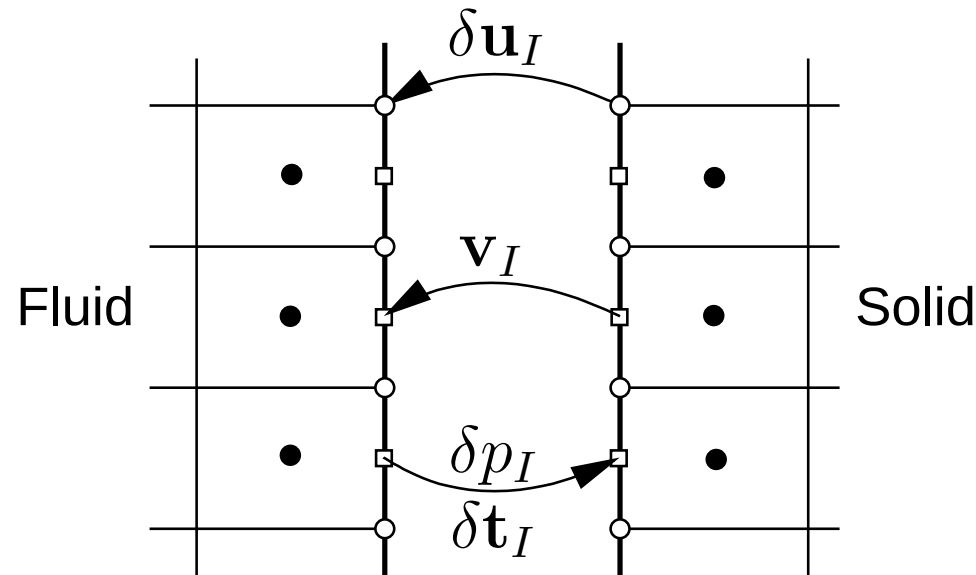
- Final form of momentum equation ready for discretisation

$$\int_{V_u} \rho_u \frac{\partial \delta \mathbf{v}}{\partial t} dV_u - \oint_{S_u} \mathbf{n}_u \bullet (2\mu + \lambda) \nabla \delta \mathbf{u} dS_u = \oint_{S_u} \mathbf{n}_u \bullet \mathbf{q} dS_u$$

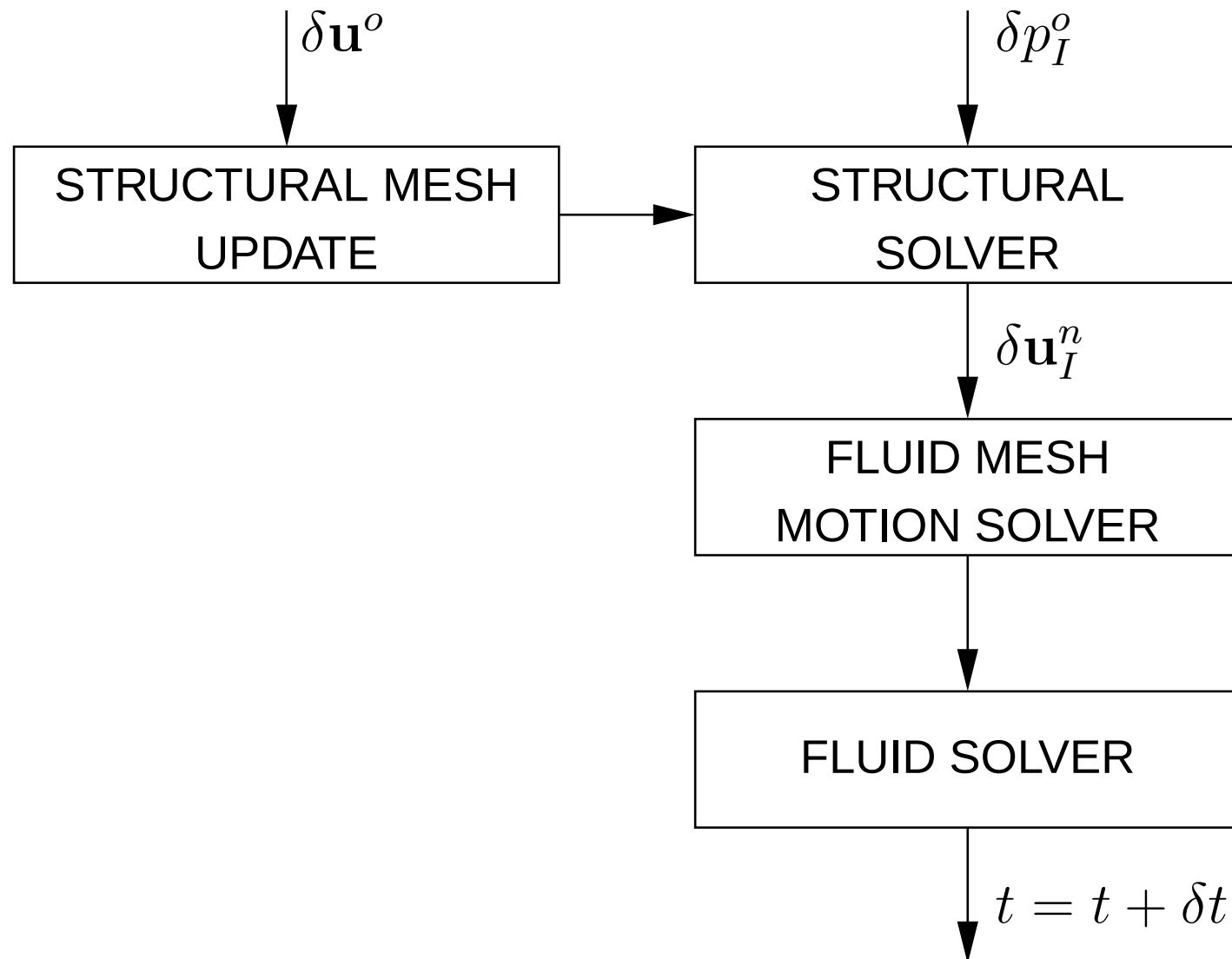
Solution Procedure

1. Update mesh according to displacement increment from the previous time step
2. Update second Piola-Kirchhoff stress tensor
3. Do until convergence
 - Calculated RHS of discretised momentum equation using last calculated displacement increment field
 - Solve momentum equation
4. Accumulate displacement vector and second Piola-Kirchhoff stress fields
5. On convergence, switch to the next time step

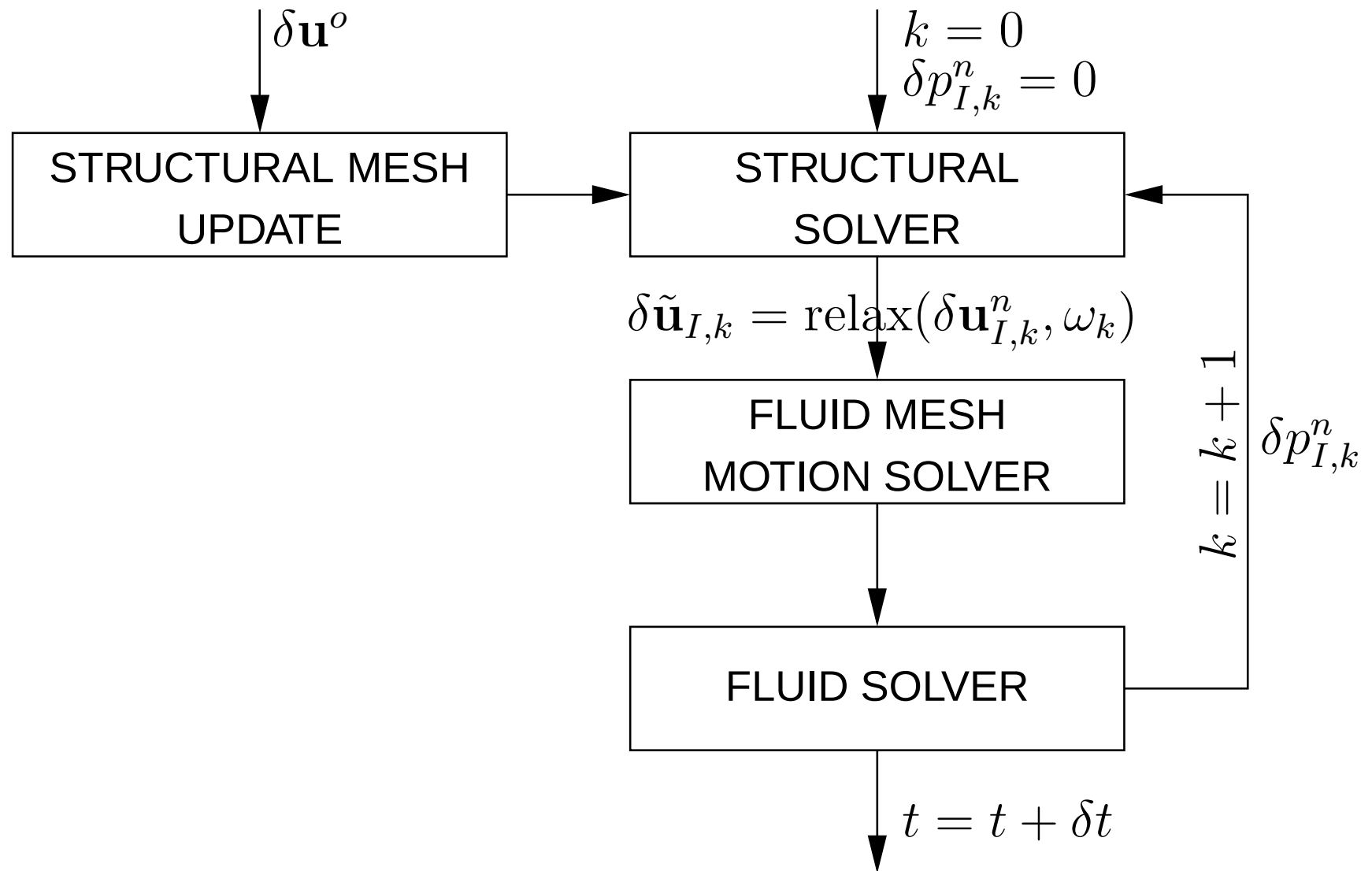
Transfer of Coupling Data



- Variables are transferred between the fluid and solid surface using patch-to-patch interpolation
- Pressure and viscous force increment (δp_I and $\delta \mathbf{t}_I$) at the fluid side of the interface is transferred to the solid side of the interface
- Displacement increment ($\delta \mathbf{u}_I$) and velocity ($\mathbf{v}_I$) at the solid side of the interface is transferred to the fluid side of the interface



Strong FSI Coupling



Adaptive Under-Relaxation Algorithm by Aitken

$$\delta \tilde{\mathbf{u}}_{I,k} = \omega_k \delta \mathbf{u}_{I,k} + (1 - \omega_k) \delta \tilde{\mathbf{u}}_{I,k-1}$$

$$\omega_k = 1 - \gamma_k$$

$$\gamma_k = \gamma_{k-1} + (\gamma_{k-1} - 1) \frac{(\Delta \delta \mathbf{u}_{k-1} - \Delta \delta \mathbf{u}_k) \cdot \Delta \delta \mathbf{u}_k}{(\Delta \delta \mathbf{u}_{k-1} - \Delta \delta \mathbf{u}_k)^2}$$

where

$$\Delta \delta \mathbf{u}_k = \delta \tilde{\mathbf{u}}_{k-1} - \delta \mathbf{u}_k$$

$$\Delta \delta \mathbf{u}_{k-1} = \delta \tilde{\mathbf{u}}_{k-2} - \delta \mathbf{u}_{k-1}$$

OpenFOAM: Object-Oriented Computational Continuum Mechanics in C++

- Natural language of continuum mechanics: partial differential equations
- Example: turbulence kinetic energy equation

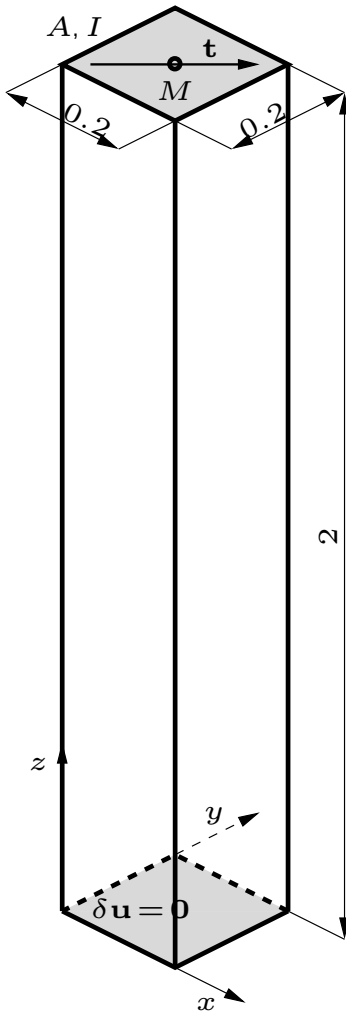
$$\frac{\partial k}{\partial t} + \nabla \cdot (\mathbf{u}k) - \nabla \cdot [(\nu + \nu_t) \nabla k] = \nu_t \left[\frac{1}{2} (\nabla \mathbf{u} + \nabla \mathbf{u}^T) \right]^2 - \frac{\epsilon_o}{k_o} k$$

- Objective: **represent differential equations in their natural language**

```
solve
(
    fvm::ddt(k)
    + fvm::div(phi, k)
    - fvm::laplacian(nu() + nut, k)
== nut*magSqr(symm(fvc::grad(U)))
    - fvm::Sp(epsilon/k, k)
);
```

- Correspondence between the implementation and the original equation is clear
- Implementation of complex physics is easy, with polyhedral mesh support, shared matrix format, efficient solvers, parallelisation and coupling tools in library form

Vibration of a 3-D Beam



Mechanical properties:

$$\rho = 1000 \frac{\text{kg}}{\text{m}^3}$$

$$E = 15.293 \times 10^6 \frac{\text{N}}{\text{m}^2}$$

$$\nu = 0.3$$

Geometric properties:

$$A = 0.04 \text{ m}^2$$

$$I = 0.0001333 \text{ m}^4$$

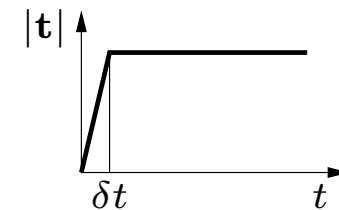
First natural frequency:

$$f_1 = 1 \text{ Hz}$$

Load:

$$\mathbf{t} = \left(\frac{|\mathbf{t}|}{\sqrt{2}}, \frac{|\mathbf{t}|}{\sqrt{2}}, 0 \right) \frac{\text{N}}{\text{m}^2}$$

$$\mathcal{F} = \frac{|\mathbf{t}|AL^2}{EI}$$

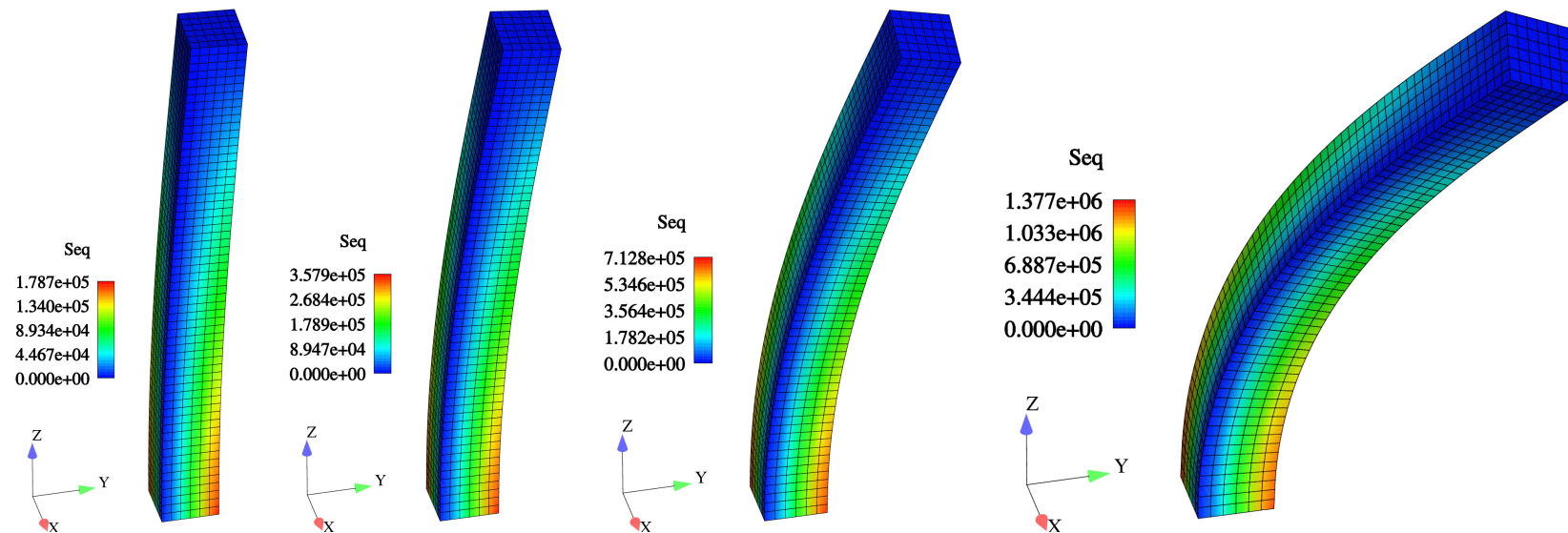


Vibration of a 3-D Beam

Static Equilibrium Solution

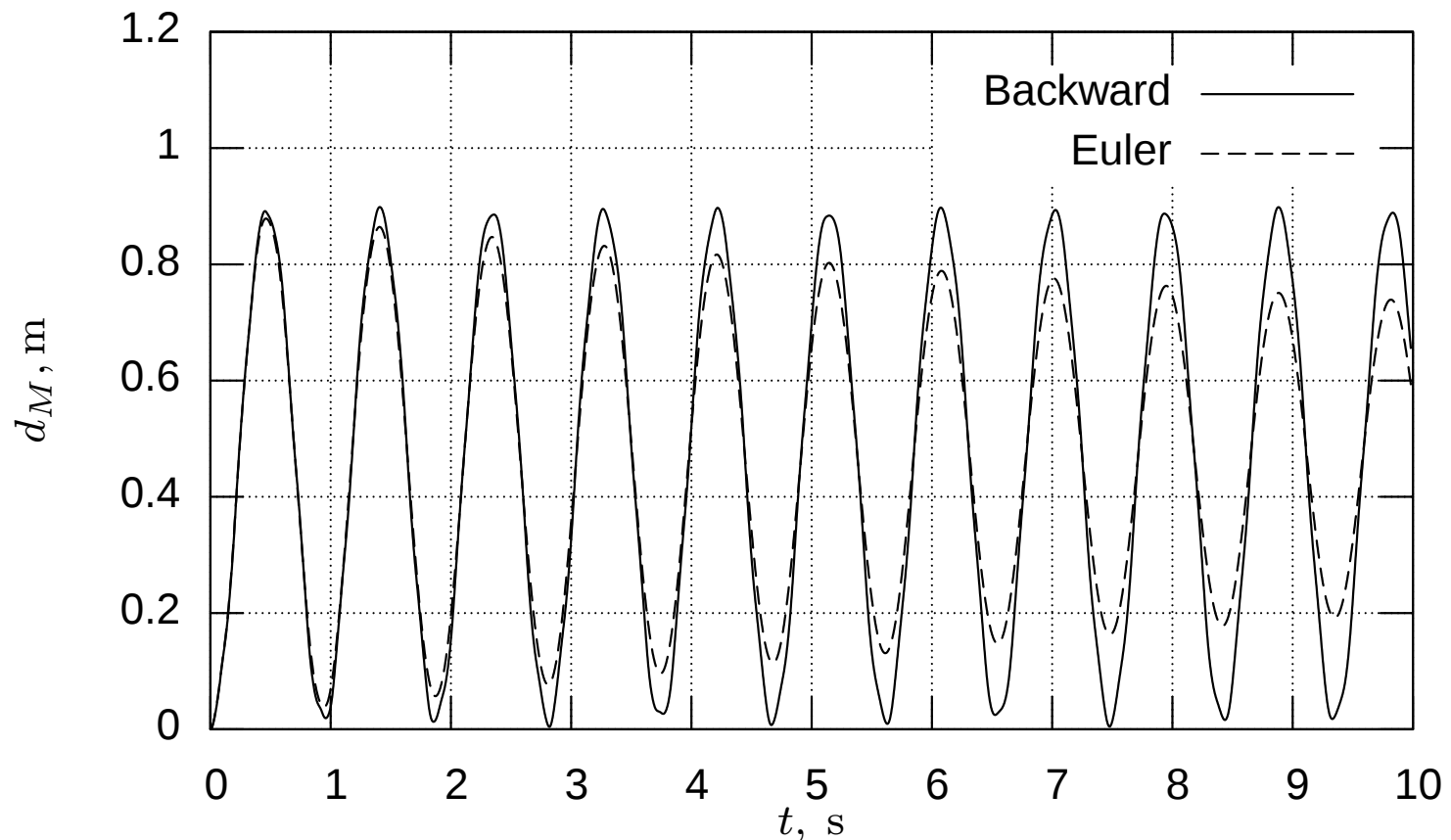
- Deflection (d_M) for various loadings ($\mathcal{F}$) and two meshes (6×60 , 10×100)

$\mathcal{F}$	d_{Ma}	$d_{M,6 \times 60}$	$E_{6 \times 60}$	$d_{M,10 \times 100}$	$E_{10 \times 100}$	Accuracy
0.2	0.13272	0.1255	5.44 %	0.1309	1.37 %	2.68
0.4	0.26196	0.2481	5.29 %	0.2581	1.47 %	2.50
0.8	0.49890	0.4797	3.85 %	0.4909	1.60 %	1.71
1.6	0.85882	0.8290	3.47 %	0.8507	0.95 %	1.50



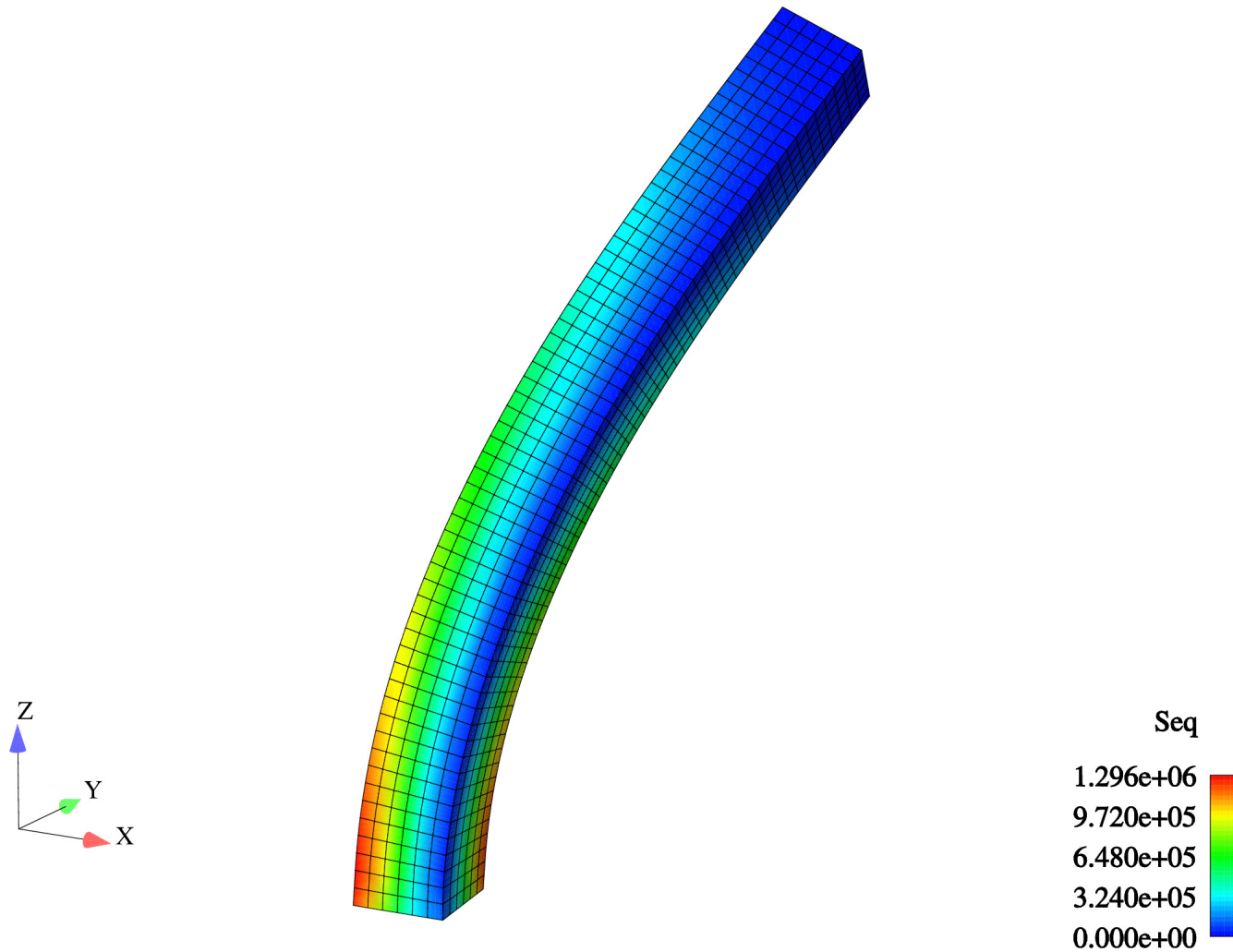
Dynamic Solution

- Tip deflection as a function of time for the Backward and Euler temporal discretisation scheme
- Time step size $\delta t = 0.002$ s corresponds to the wave $Co = 6$



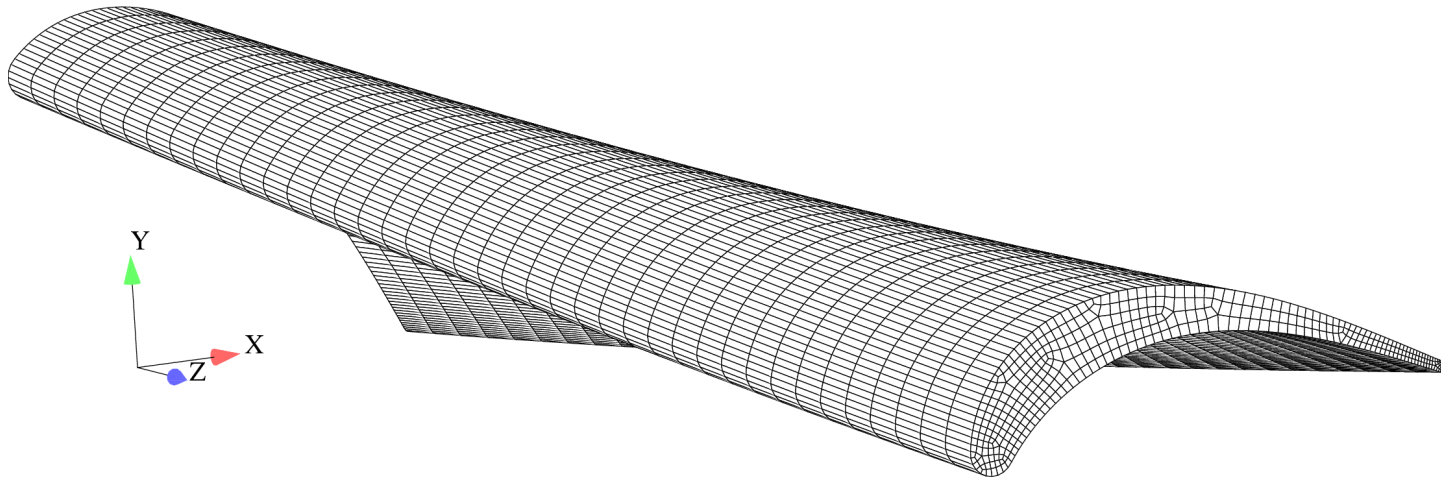
Vibration of a 3-D Beam

Beam Vibration: Linear Elastic Material with Large Deformations



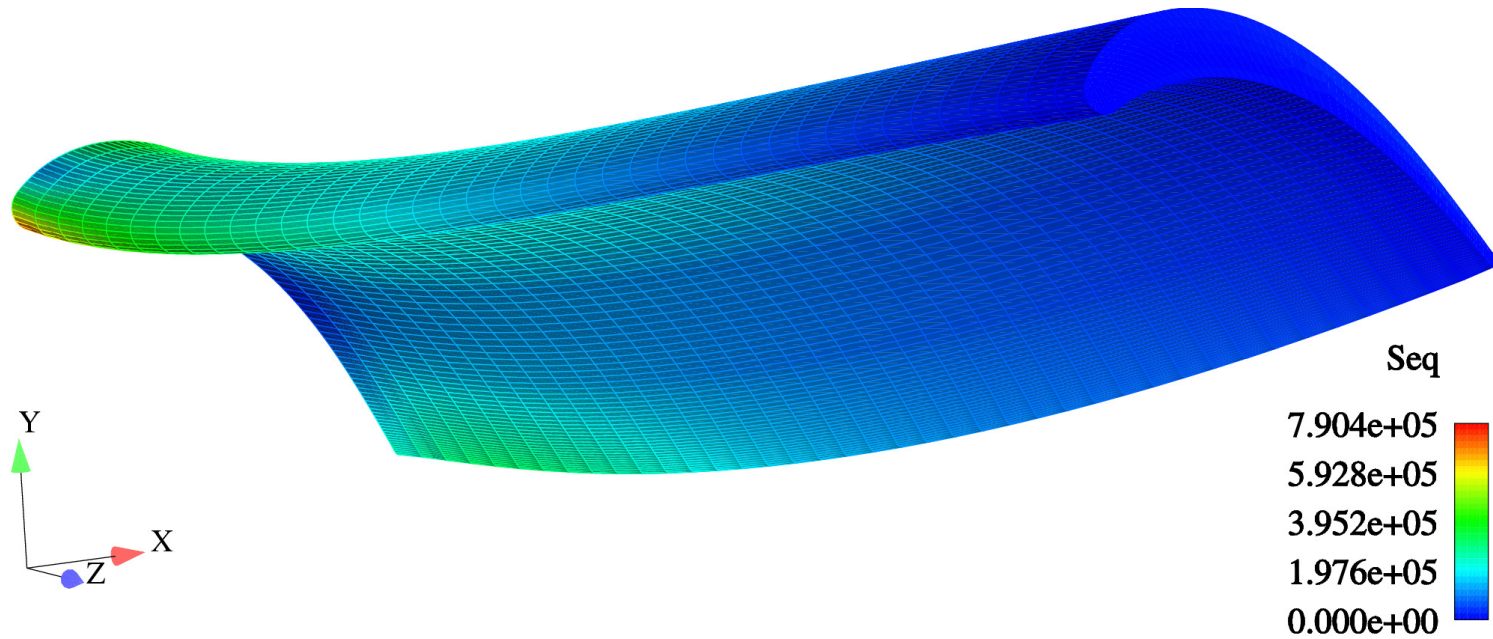
Vibration on an Unstructured Mesh

- Twisted axial turbine blade, 0.8 m in height, 0.2 m cord
- Mechanical properties are the same as of the 3-D square beam
- The unstructured hexahedral mesh consists of 50000 cells
- The blade is constrained at one end and subjected to a suddenly applied traction force at the other end



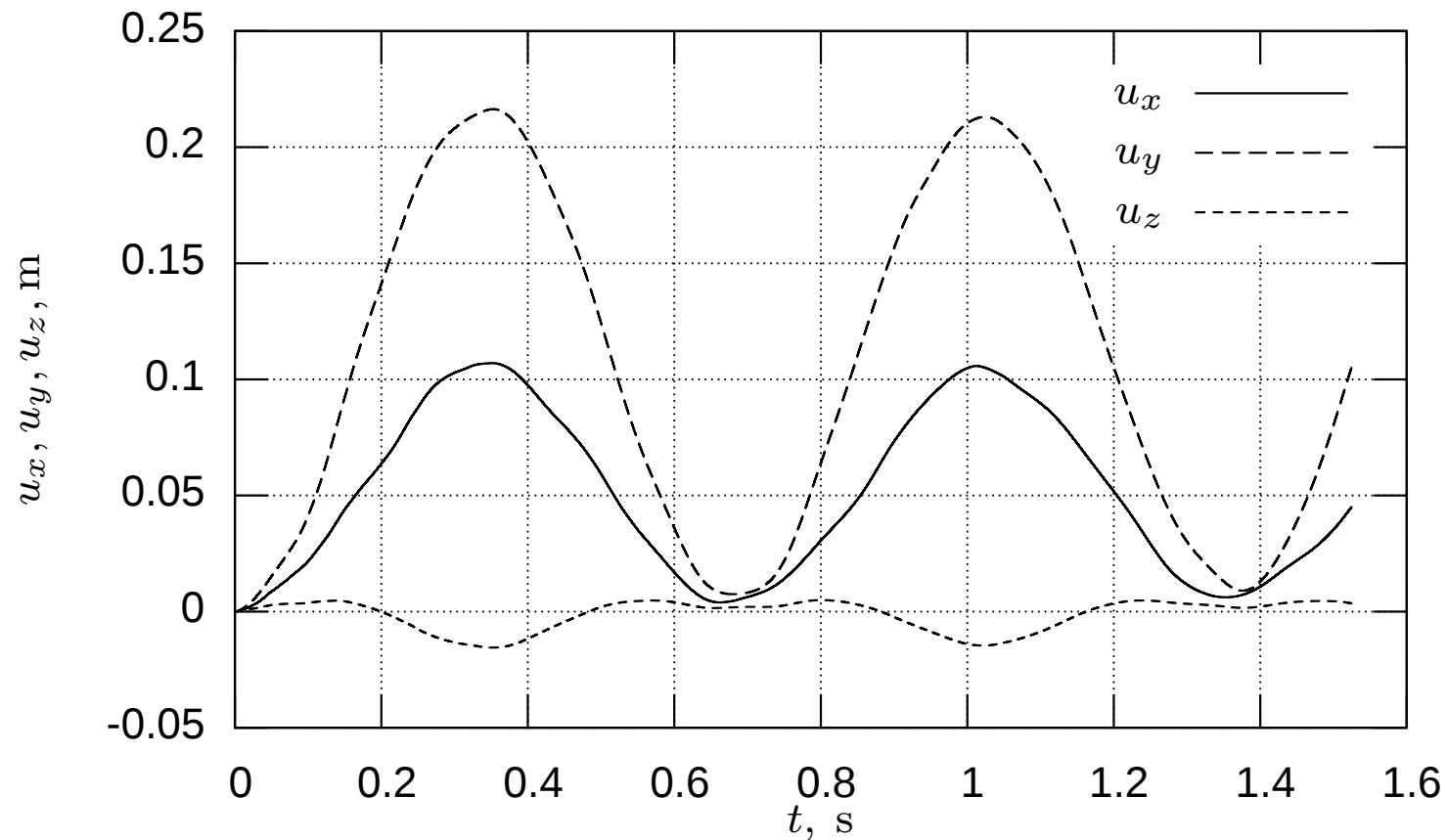
Vibration of a Turbine Blade

Mean Equilibrium State

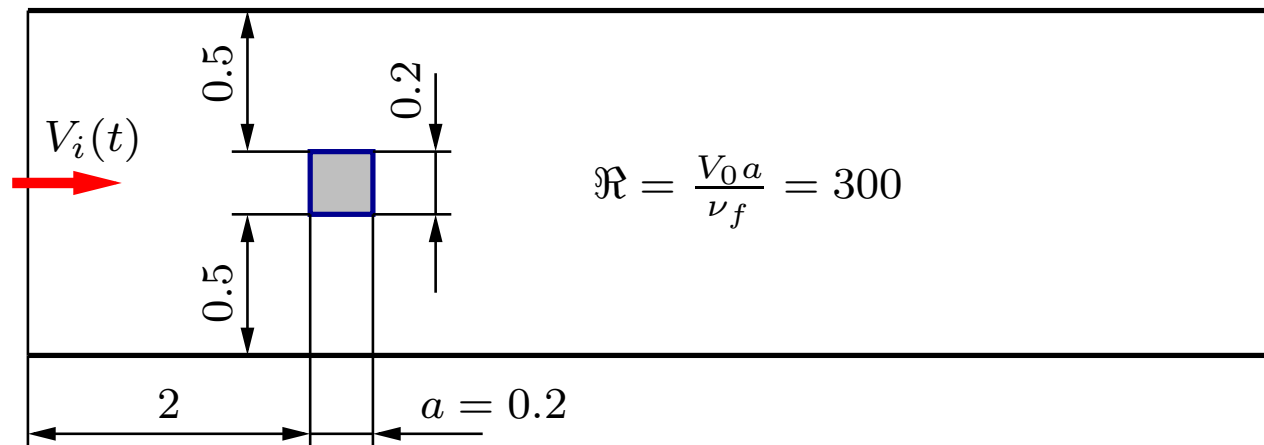
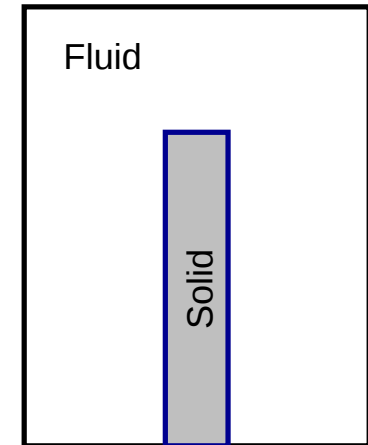
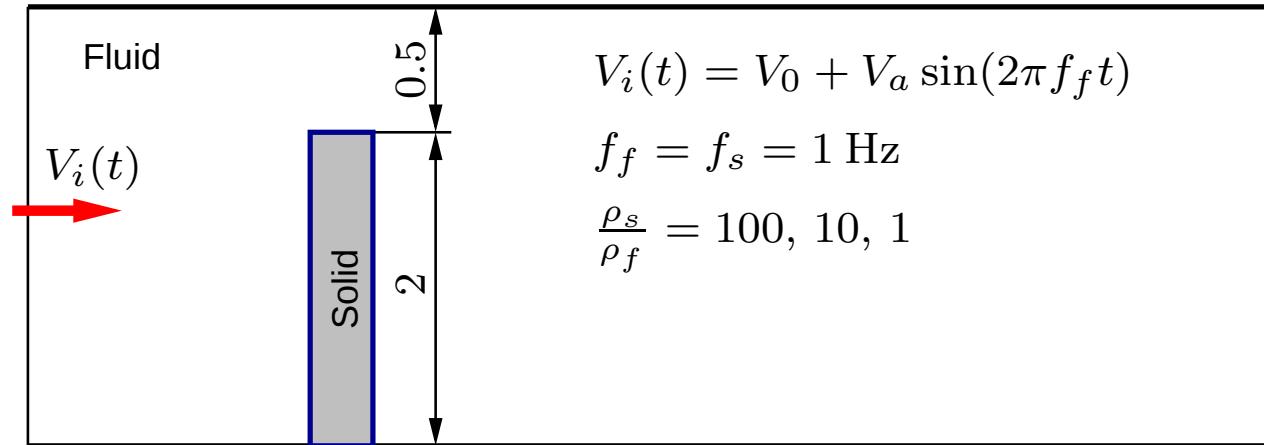


Vibration of a Turbine Blade

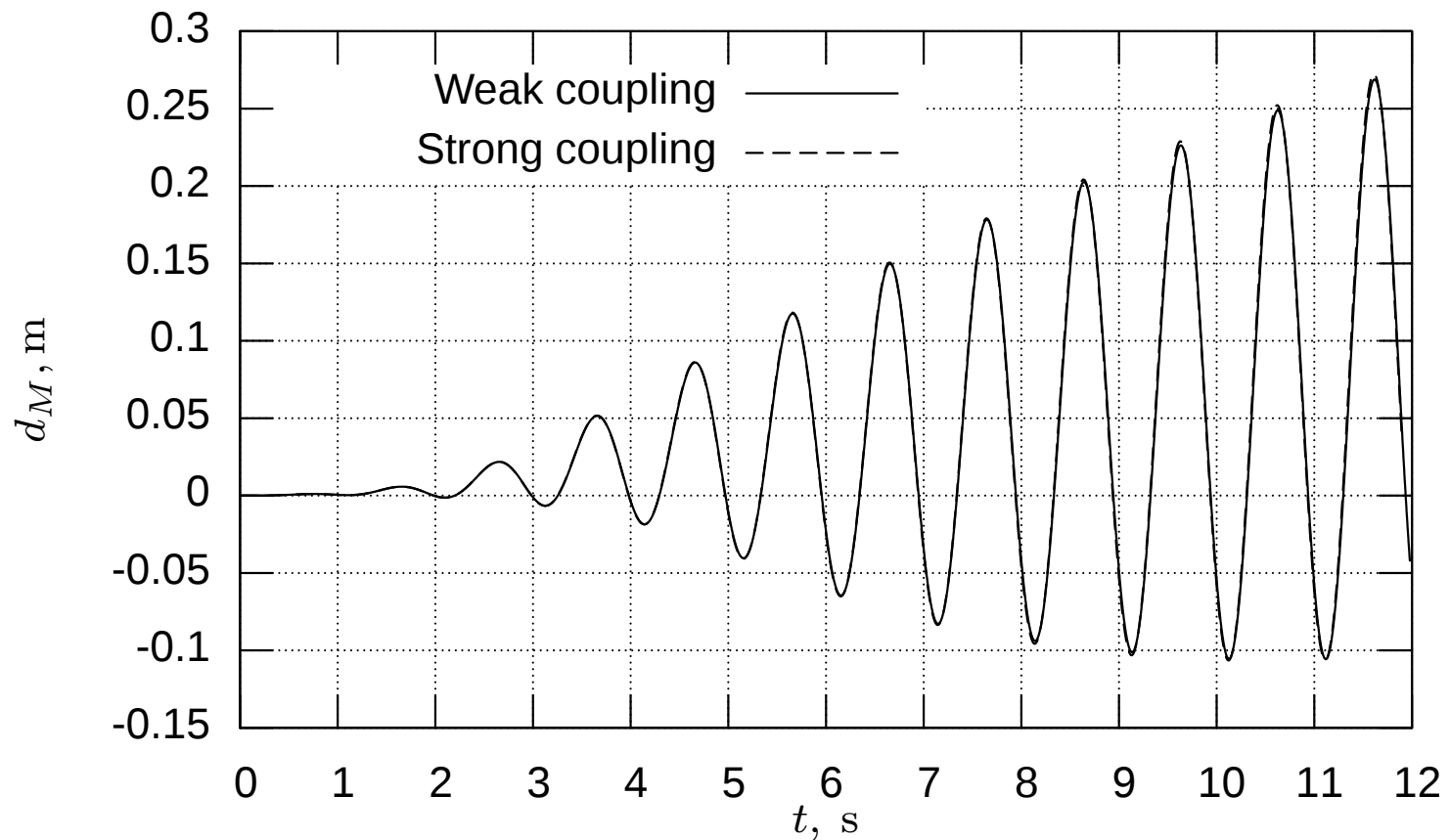
Dynamic Response: displacement components of the top of leading edge



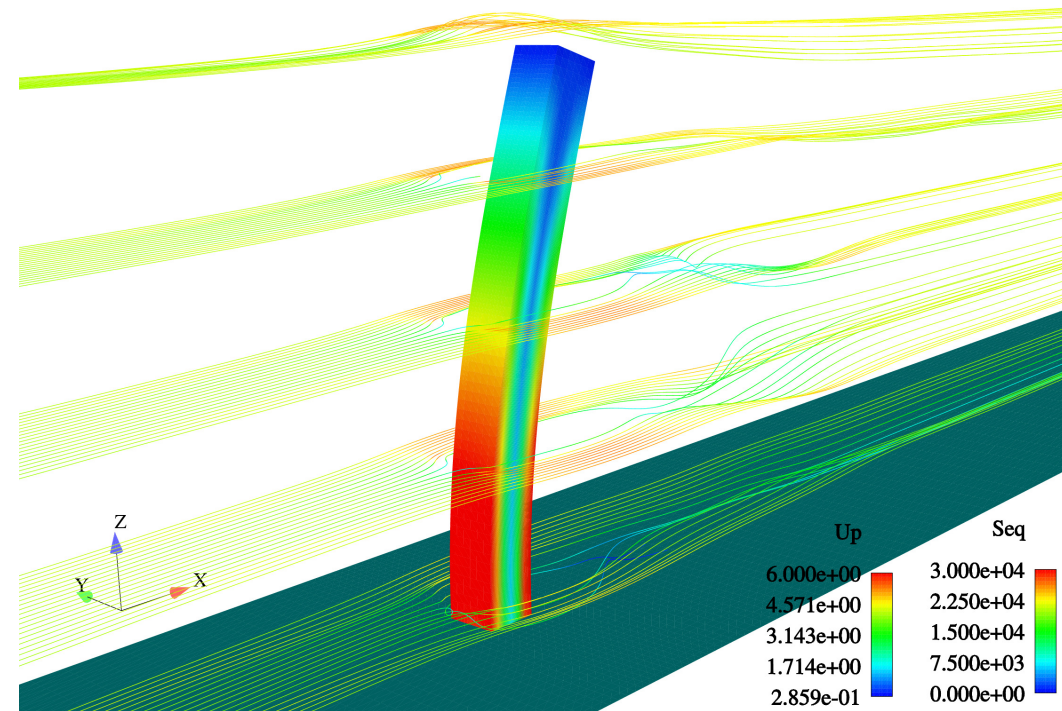
Flow-Induced Vibration: Problem Description



Flow-Induced Vibration: Solution for Limiting Density Ratio $\rho_s/\rho_f = 100$



Cantilevered Beam Vibration in an Oscillating Flow Field



Effect of Under-Relaxation

ρ_s / ρ_f	Number of outer iterations per time-step	
	Fixed under-relaxation	Adaptive under-relaxation
10	30	6
1	60	12

Summary

- Self-contained FVM FSI solver implemented in OpenFOAM
 - Fluid flow solver with run-time selection of physics
 - Structural analysis: updated Lagrangian formulation, non-linear materials
 - OpenFOAM provides surface coupling utilities: patch-to-patch interpolation
- OpenFOAM is a good platform for a close-coupled FSI: extensive capabilities, easy physics model implementation, code sharing, mapping tools
- The solver shows reasonable efficiency for weak fluid-structure interaction
- Strong coupling algorithm with Aitken adaptive under-relaxation shows considerable improvement in comparison with fixed under-relaxation

Future Work

- Development of a coupled solver in OpenFOAM (work in progress) will considerably improve the efficiency of structural and FSI solver: **shared matrix classes** and block-matrix solution algorithm
- Further convergence acceleration for simulation of strong FSI can be achieved by using **Reduced Order Model (ROM)** for fluid and/or solid: a POD-based ROM will be used to improve convergence of the strong coupling algorithm
- Inclusion of **contact stresses and crack propagation** models for structures